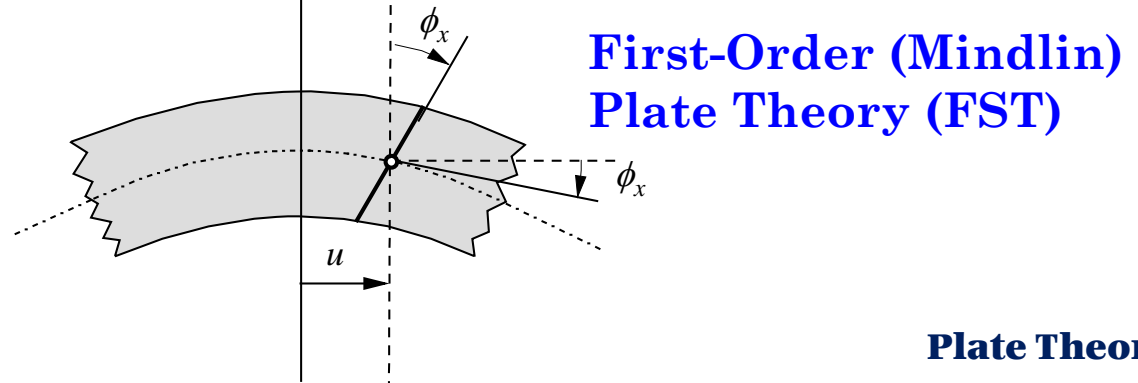
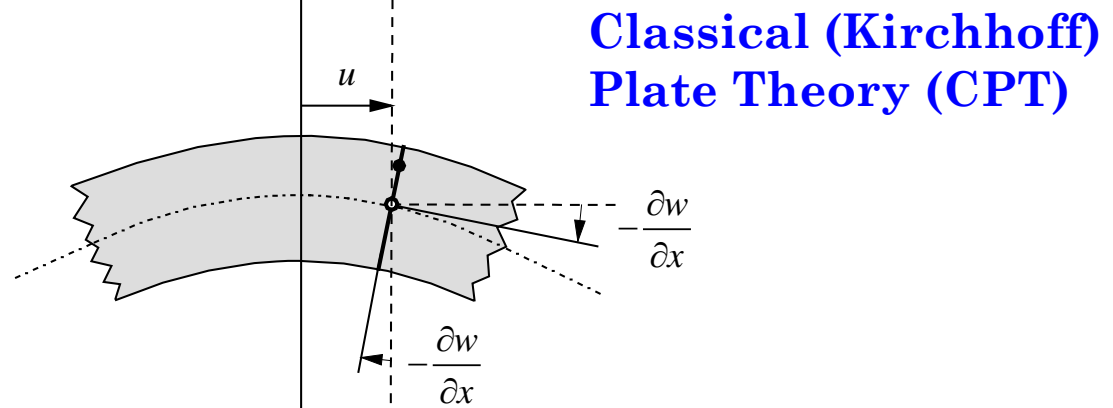
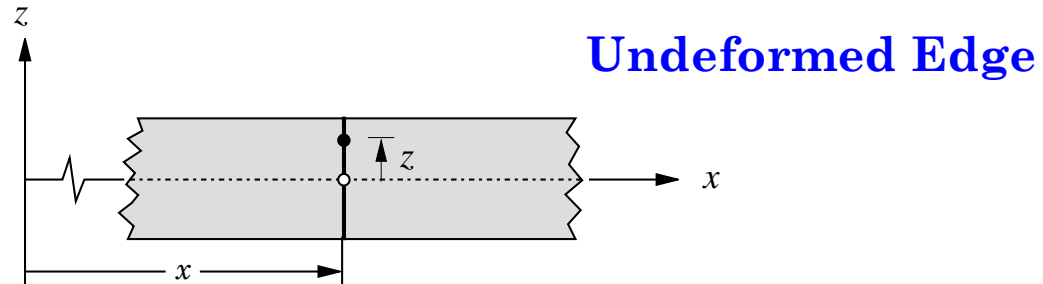
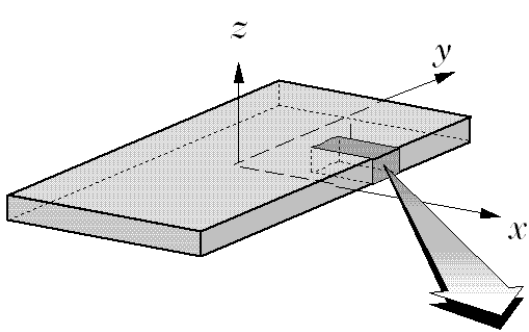




**LAMINATE PLATE THEORIES
AND ANALYTICAL SOLUTIONS
OF RECTANGULAR PLATES**

Bending, Vibration, **Buckling**

Kinematics of the Classical and Shear Deformation Plate Theories





Classical Laminate Plate Theory (CLPT)

$$u(x, y, z, t) = u_0(x, y, t) - z \frac{\partial w_0}{\partial x}$$

$$v(x, y, z, t) = v_0(x, y, t) - z \frac{\partial w_0}{\partial y}$$

$$w(x, y, z, t) = w_0(x, y, t)$$

First-order Shear Deformation Theory (FSDT)

$$u(x, y, z, t) = u_0(x, y, t) + z\phi_x(x, y, t)$$

$$v(x, y, z, t) = v_0(x, y, t) + z\phi_y(x, y, t)$$

$$w(x, y, z, t) = w_0(x, y, t)$$

Third-order Shear Deformation Theory (TSDT)

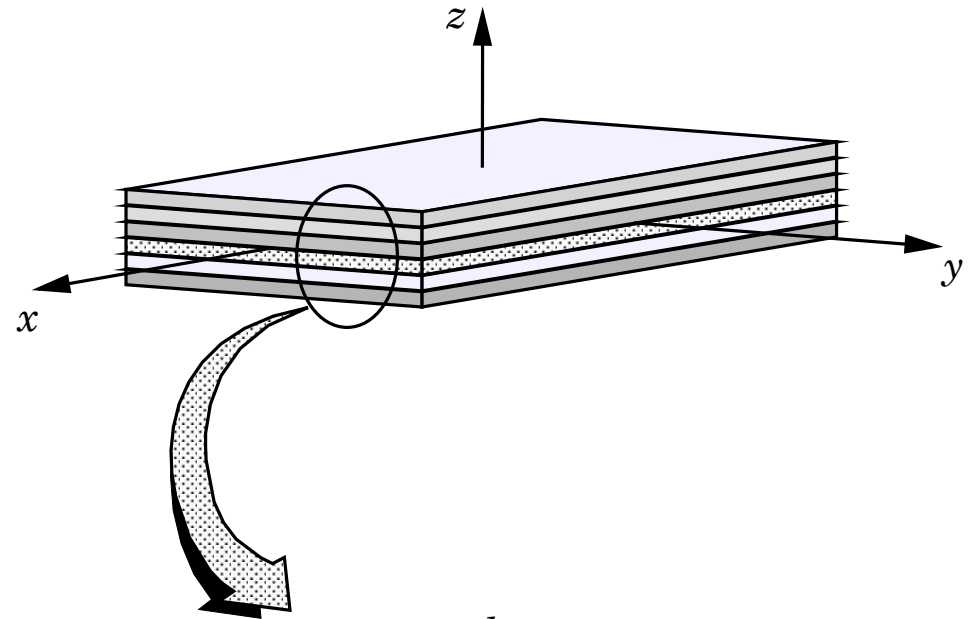
$$u(x, y, z, t) = u_0(x, y, t) + z\phi_x(x, y, t) + z^3 \left(-\frac{4}{3h^2} \right) \left(\phi_x + \frac{\partial w_0}{\partial x} \right)$$

$$v(x, y, z, t) = v_0(x, y, t) + z\phi_y(x, y, t) + z^3 \left(-\frac{4}{3h^2} \right) \left(\phi_y + \frac{\partial w_0}{\partial y} \right)$$

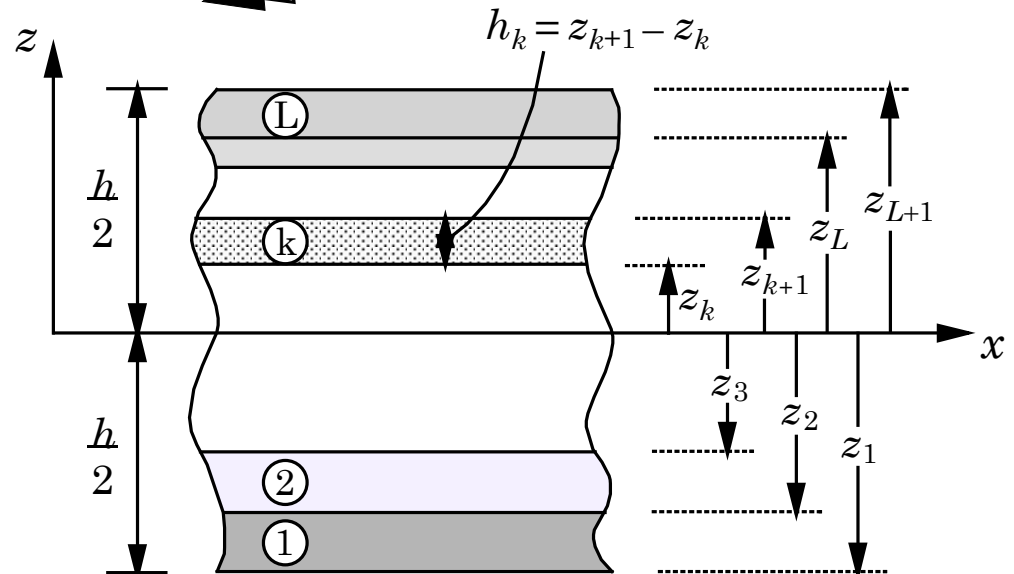
$$w(x, y, z, t) = w_0(x, y, t)$$

Coordinate System and Lamination Scheme

Notation and Coordinate System Used in a Laminate Analysis



Layers are numbered in the +ve z direction



Displacement Field of CLPT

$$u(x, y, z, t) = u_0(x, y, t) - z \frac{\partial w_0}{\partial x}$$

$$v(x, y, z, t) = v_0(x, y, t) - z \frac{\partial w_0}{\partial y}$$

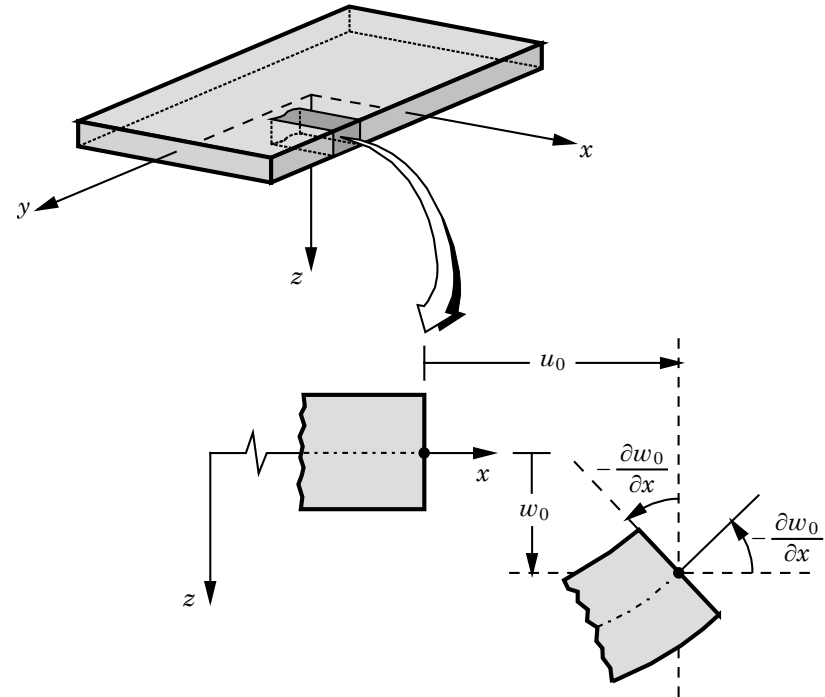
$$w(x, y, z, t) = w_0(x, y, t)$$

Strains

$$\begin{Bmatrix} \varepsilon_{xx} \\ \varepsilon_{yy} \\ \gamma_{xy} \end{Bmatrix} = \begin{Bmatrix} \varepsilon_{xx}^{(0)} \\ \varepsilon_{yy}^{(0)} \\ \gamma_{xy}^{(0)} \end{Bmatrix} + z \begin{Bmatrix} \varepsilon_{xx}^{(1)} \\ \varepsilon_{yy}^{(1)} \\ \gamma_{xy}^{(1)} \end{Bmatrix}$$

$$\{\varepsilon^0\} = \begin{Bmatrix} \varepsilon_{xx}^{(0)} \\ \varepsilon_{yy}^{(0)} \\ \gamma_{xy}^{(0)} \end{Bmatrix} = \begin{Bmatrix} \frac{\partial u_0}{\partial x} + \frac{1}{2} \left(\frac{\partial w_0}{\partial x} \right)^2 \\ \frac{\partial v_0}{\partial y} + \frac{1}{2} \left(\frac{\partial w_0}{\partial y} \right)^2 \\ \frac{\partial u_0}{\partial y} + \frac{\partial v_0}{\partial x} + \frac{\partial w_0}{\partial x} \frac{\partial w_0}{\partial y} \end{Bmatrix}$$

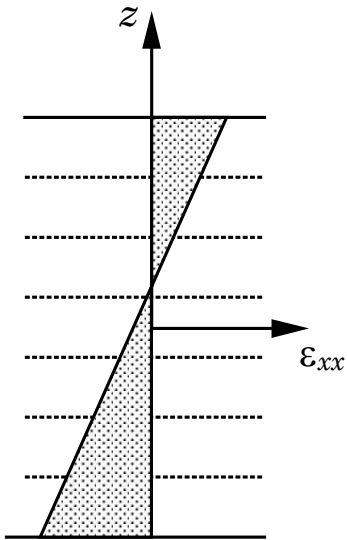
$$\{\varepsilon^1\} = \begin{Bmatrix} \varepsilon_{xx}^{(1)} \\ \varepsilon_{yy}^{(1)} \\ \gamma_{xy}^{(1)} \end{Bmatrix} = \begin{Bmatrix} -\frac{\partial^2 w_0}{\partial x^2} \\ -\frac{\partial^2 w_0}{\partial y^2} \\ -2 \frac{\partial^2 w_0}{\partial x \partial y} \end{Bmatrix}$$



Lamina Constitutive Relations

$$\begin{Bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{xy} \end{Bmatrix}^{(k)} = \begin{bmatrix} \bar{Q}_{11} & \bar{Q}_{12} & \bar{Q}_{16} \\ \bar{Q}_{12} & \bar{Q}_{22} & \bar{Q}_{26} \\ \bar{Q}_{16} & \bar{Q}_{26} & \bar{Q}_{66} \end{bmatrix}^{(k)} \left(\begin{Bmatrix} \varepsilon_{xx} \\ \varepsilon_{yy} \\ \gamma_{xy} \end{Bmatrix} - \begin{Bmatrix} \alpha_{xx} \\ \alpha_{yy} \\ 2\alpha_{xy} \end{Bmatrix} \Delta T \right)$$

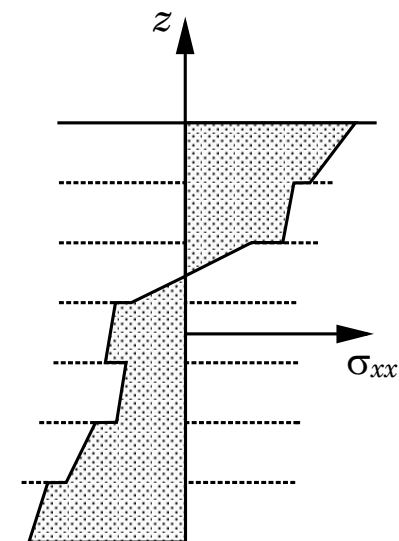
Strain and Stress Distributions Through the Thickness of the Laminate



Strain distribution is continuous through laminate thickness

$$\begin{Bmatrix} \varepsilon_{xx} \\ \varepsilon_{yy} \\ \gamma_{xy} \end{Bmatrix} = \begin{Bmatrix} \varepsilon_{xx}^{(0)} \\ \varepsilon_{yy}^{(0)} \\ \gamma_{xy}^{(0)} \end{Bmatrix} + z \begin{Bmatrix} \varepsilon_{xx}^{(1)} \\ \varepsilon_{yy}^{(1)} \\ \gamma_{xy}^{(1)} \end{Bmatrix}$$

Stress distribution is discontinuous through laminate thickness



$$\begin{Bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{xy} \end{Bmatrix}^{(k)} = \begin{bmatrix} \bar{Q}_{11} & \bar{Q}_{12} & \bar{Q}_{16} \\ \bar{Q}_{12} & \bar{Q}_{22} & \bar{Q}_{26} \\ \bar{Q}_{16} & \bar{Q}_{26} & \bar{Q}_{66} \end{bmatrix}^{(k)} \begin{Bmatrix} \varepsilon_{xx} \\ \varepsilon_{yy} \\ \gamma_{xy} \end{Bmatrix}$$

$$\begin{Bmatrix} \sigma_{yz} \\ \sigma_{xz} \end{Bmatrix}^{(k)} = \begin{bmatrix} \bar{Q}_{44} & \bar{Q}_{45} \\ \bar{Q}_{45} & \bar{Q}_{55} \end{bmatrix}^{(k)} \begin{Bmatrix} \gamma_{yz} \\ \gamma_{xz} \end{Bmatrix}$$

CLASSICAL LAMINATE THEORY (continued)

Principle of Virtual Displacements

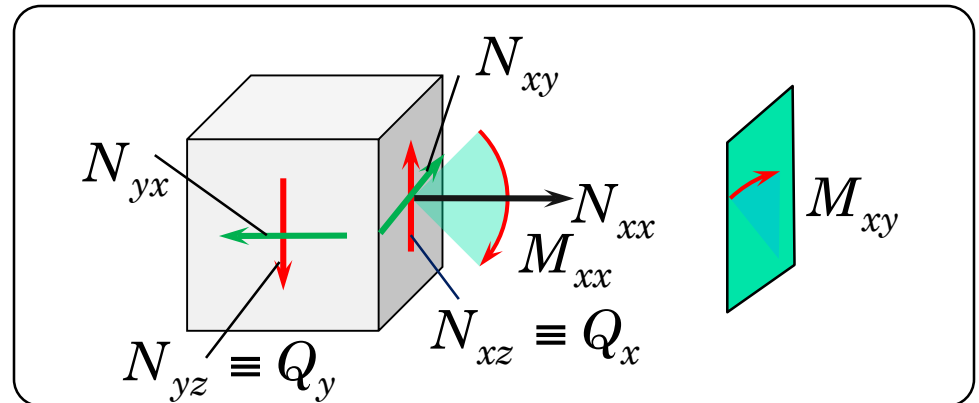
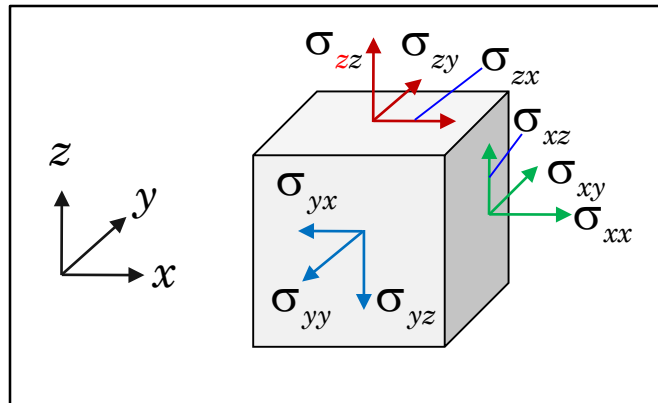
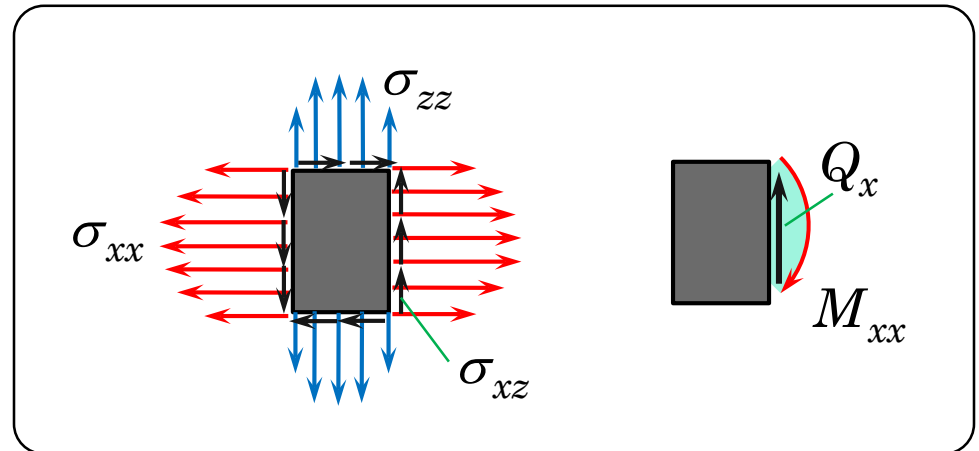
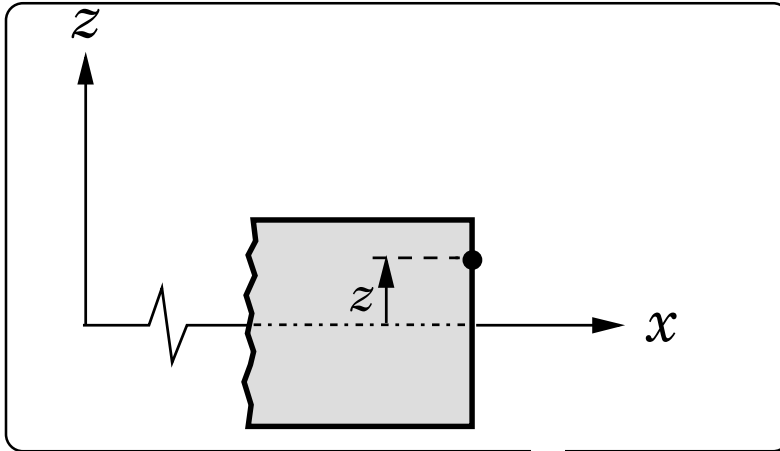
$$\begin{aligned}
 0 &= \int_{V^e} \sigma_{ij} \delta \varepsilon_{ij} dV - \int_{\Omega^e} q \delta w dxdy - \int_{\Gamma^e} (V_n \delta w + M_n \delta \theta_n) ds \\
 &= \int_{\Omega^e} \int_{-h/2}^{h/2} (\sigma_{xx} \delta \varepsilon_{xx} + \sigma_{yy} \delta \varepsilon_{yy} + \sigma_{xy} 2\delta \varepsilon_{xy}) dz dxdy \\
 &\quad - \int_{\Omega^e} q \delta w_0 dxdy - \int_{\Gamma^e} (N_n \delta u_n + N_{ns} \delta u_{ns} + V_n \delta w_0 + M_n \delta \theta_n) ds \\
 &= \int_{\Omega^e} \left\{ \int_{-h/2}^{h/2} \left[\sigma_{xx} \left(\frac{\partial \delta u_0}{\partial x} + \frac{\partial w_0}{\partial x} \frac{\partial \delta w_0}{\partial x} - z \frac{\partial^2 \delta w_0}{\partial x^2} \right) + \sigma_{yy} \left(\frac{\partial \delta v_0}{\partial y} + \frac{\partial w_0}{\partial y} \frac{\partial \delta w_0}{\partial y} - z \frac{\partial^2 \delta w_0}{\partial y^2} \right) \right. \right. \\
 &\quad \left. \left. + \sigma_{xy} \left(\frac{\partial \delta u_0}{\partial y} + \frac{\partial \delta v_0}{\partial x} + \frac{\partial w_0}{\partial x} \frac{\partial \delta w_0}{\partial y} + \frac{\partial w_0}{\partial y} \frac{\partial \delta w_0}{\partial x} - 2z \frac{\partial^2 \delta w_0}{\partial x \partial y} \right) \right] dz \right\} dxdy \\
 &\quad - \int_{\Omega^e} q \delta w_0 dxdy - \int_{\Gamma^e} (N_n \delta u_n + N_{ns} \delta u_{ns} + V_n \delta w_0 + M_n \delta \theta_n) ds
 \end{aligned}$$

CLASSICAL LAMINATE THEORY (continued)

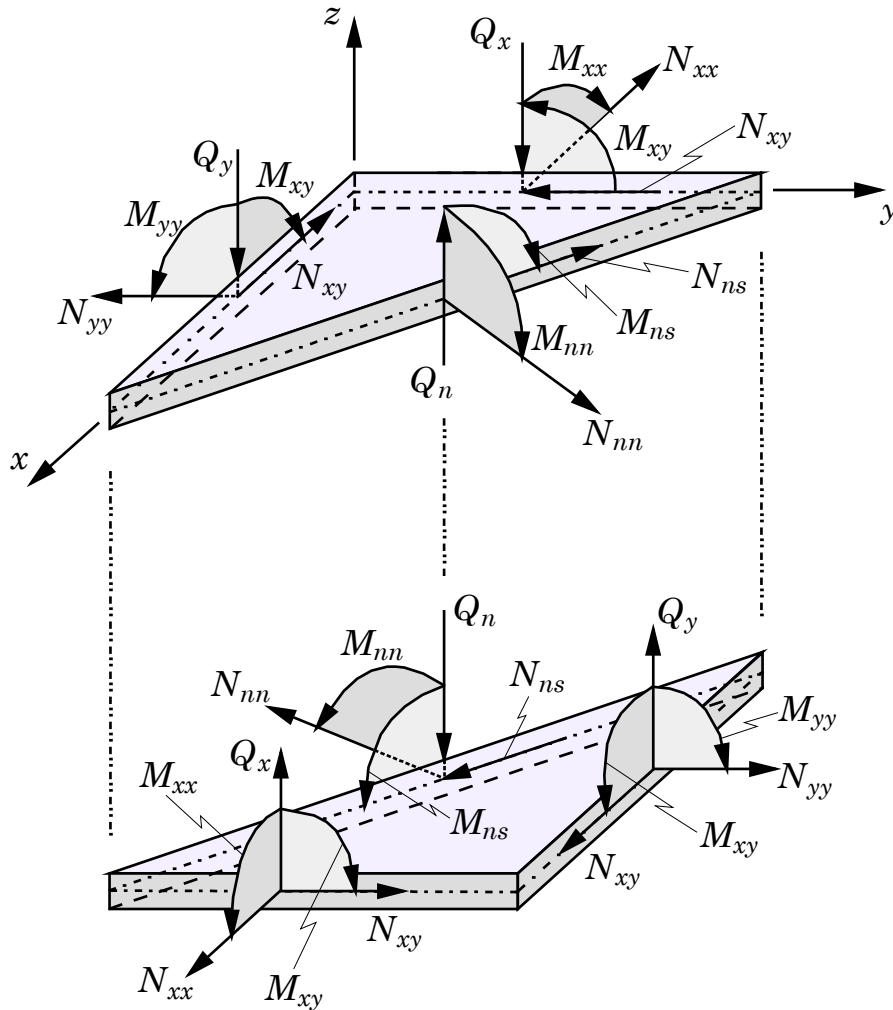
Principle of Virtual Displacements (continued)

$$\begin{aligned} 0 = & \int_{\Omega^e} \left\{ N_{xx} \left(\frac{\partial \delta u_0}{\partial x} + \frac{\partial w_0}{\partial x} \frac{\partial \delta w_0}{\partial x} \right) + N_{yy} \left(\frac{\partial \delta v_0}{\partial y} + \frac{\partial w_0}{\partial y} \frac{\partial \delta w_0}{\partial y} \right) \right. \\ & + N_{xy} \left(\frac{\partial \delta u_0}{\partial y} + \frac{\partial \delta v_0}{\partial x} + \frac{\partial w_0}{\partial x} \frac{\partial \delta w_0}{\partial y} + \frac{\partial w_0}{\partial y} \frac{\partial \delta w_0}{\partial x} \right) \\ & \left. - M_{xx} \frac{\partial^2 \delta w_0}{\partial x^2} - M_{yy} \frac{\partial^2 \delta w_0}{\partial y^2} - 2M_{xy} \frac{\partial^2 \delta w_0}{\partial x \partial y} \right\} dx dy \\ & - \int_{\Omega^e} q \delta w_0 dx dy - \int_{\Gamma^e} (N_n \delta u_n + N_{ns} \delta u_{ns} + V_n \delta w_0 + M_n \delta \theta_n) ds \end{aligned}$$

SIGN CONVENTION FOR STRESS RESULTANTS



STRESS RESULTANTS



$$N_{\xi\eta} = \sum_{k=1}^N \int_{z_k}^{z_{k+1}} \sigma_{\xi\eta} dz \quad (\xi, \eta = x, y, z)$$

$$M_{\xi\eta} = \sum_{k=1}^N \int_{z_k}^{z_{k+1}} \sigma_{\xi\eta} z dz \quad (\xi, \eta = x, y)$$

$$Q_x = N_{xz}, \quad Q_y = N_{yz}$$

Equations of Motion of the Classical Plate Theory

$$\begin{aligned}
 \begin{Bmatrix} M_{xx} \\ M_{yy} \\ M_{xy} \end{Bmatrix} &= \sum_{k=1}^N \int_{z_k}^{z_{k+1}} \begin{Bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{xy} \end{Bmatrix} z \, dz \\
 &= \sum_{k=1}^N \int_{z_k}^{z_{k+1}} \begin{bmatrix} \bar{Q}_{11} & \bar{Q}_{12} & \bar{Q}_{16} \\ \bar{Q}_{12} & \bar{Q}_{22} & \bar{Q}_{26} \\ \bar{Q}_{16} & \bar{Q}_{26} & \bar{Q}_{66} \end{bmatrix}^{(k)} \begin{Bmatrix} \varepsilon_{xx}^{(0)} + z\varepsilon_{xx}^{(1)} \\ \varepsilon_{yy}^{(0)} + z\varepsilon_{yy}^{(1)} \\ \gamma_{xy}^{(0)} + z\gamma_{xy}^{(1)} \end{Bmatrix} z \, dz \\
 \begin{Bmatrix} M_{xx} \\ M_{yy} \\ M_{xy} \end{Bmatrix} &= \begin{bmatrix} B_{11} & B_{12} & B_{16} \\ B_{12} & B_{22} & B_{26} \\ B_{16} & B_{26} & B_{66} \end{bmatrix} \begin{Bmatrix} \varepsilon_{xx}^{(0)} \\ \varepsilon_{yy}^{(0)} \\ \gamma_{xy}^{(0)} \end{Bmatrix} + \begin{bmatrix} D_{11} & D_{12} & D_{16} \\ D_{12} & D_{22} & D_{26} \\ D_{16} & D_{26} & D_{66} \end{bmatrix} \begin{Bmatrix} \varepsilon_{xx}^{(1)} \\ \varepsilon_{yy}^{(1)} \\ \gamma_{xy}^{(1)} \end{Bmatrix}
 \end{aligned}$$

Laminate Stiffnesses

$$(A_{ij}, B_{ij}, D_{ij}) = \int_{-\frac{h}{2}}^{\frac{h}{2}} \bar{Q}_{ij}(1, z, z^2) dz = \sum_{k=1}^N \int_{z_k}^{z_{k+1}} \bar{Q}_{ij}^{(k)}(1, z, z^2) dz$$

$$A_{ij} = \sum_{k=1}^N \bar{Q}_{ij}^{(k)} (z_{k+1} - z_k), \quad B_{ij} = \frac{1}{2} \sum_{k=1}^N \bar{Q}_{ij}^{(k)} (z_{k+1}^2 - z_k^2)$$

$$D_{ij} = \frac{1}{3} \sum_{k=1}^N \bar{Q}_{ij}^{(k)} (z_{k+1}^3 - z_k^3)$$

Equations of Motion of the Classical Plate Theory

$$\begin{aligned} \delta u_0 : \quad & \frac{\partial N_{xx}}{\partial x} + \frac{\partial N_{xy}}{\partial y} = I_0 \frac{\partial^2 u_0}{\partial t^2} - I_1 \frac{\partial^2}{\partial t^2} \left(\frac{\partial w_0}{\partial x} \right) \\ \delta v_0 : \quad & \frac{\partial N_{xy}}{\partial x} + \frac{\partial N_{yy}}{\partial y} = I_0 \frac{\partial^2 v_0}{\partial t^2} - I_1 \frac{\partial^2}{\partial t^2} \left(\frac{\partial w_0}{\partial y} \right) \\ \delta w_0 : \quad & \frac{\partial^2 M_{xx}}{\partial x^2} + 2 \frac{\partial^2 M_{xy}}{\partial y \partial x} + \frac{\partial^2 M_{yy}}{\partial y^2} + \mathcal{N}(w_0) + q = I_0 \frac{\partial^2 w_0}{\partial t^2} \\ & - I_2 \frac{\partial^2}{\partial t^2} \left(\frac{\partial^2 w_0}{\partial x^2} + \frac{\partial^2 w_0}{\partial y^2} \right) + I_1 \frac{\partial^2}{\partial t^2} \left(\frac{\partial u_0}{\partial x} + \frac{\partial v_0}{\partial y} \right) \end{aligned}$$

$$\begin{aligned} \begin{Bmatrix} N_{xx} \\ N_{yy} \\ N_{xy} \end{Bmatrix} &= \sum_{k=1}^N \int_{z_k}^{z_{k+1}} \begin{Bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{xy} \end{Bmatrix} dz \\ &= \sum_{k=1}^N \int_{z_k}^{z_{k+1}} \begin{bmatrix} \bar{Q}_{11} & \bar{Q}_{12} & \bar{Q}_{16} \\ \bar{Q}_{12} & \bar{Q}_{22} & \bar{Q}_{26} \\ \bar{Q}_{16} & \bar{Q}_{26} & \bar{Q}_{66} \end{bmatrix}^{(k)} \begin{Bmatrix} \varepsilon_{xx}^{(0)} + z\varepsilon_{xx}^{(1)} \\ \varepsilon_{yy}^{(0)} + z\varepsilon_{yy}^{(1)} \\ \gamma_{xy}^{(0)} + z\gamma_{xy}^{(1)} \end{Bmatrix} dz \\ \begin{Bmatrix} N_{xx} \\ N_{yy} \\ N_{xy} \end{Bmatrix} &= \begin{bmatrix} A_{11} & A_{12} & A_{16} \\ A_{12} & A_{22} & A_{26} \\ A_{16} & A_{26} & A_{66} \end{bmatrix} \begin{Bmatrix} \varepsilon_{xx}^{(0)} \\ \varepsilon_{yy}^{(0)} \\ \gamma_{xy}^{(0)} \end{Bmatrix} + \begin{bmatrix} B_{11} & B_{12} & B_{16} \\ B_{12} & B_{22} & B_{26} \\ B_{16} & B_{26} & B_{66} \end{bmatrix} \begin{Bmatrix} \varepsilon_{xx}^{(1)} \\ \varepsilon_{yy}^{(1)} \\ \gamma_{xy}^{(1)} \end{Bmatrix} \end{aligned}$$

Stress Resultant-Displacement Relations

$$\begin{aligned}
 N_{xx} &= \int_{-h/2}^{h/2} \sigma_{xx} dz = \int_{-h/2}^{h/2} \left\{ \bar{Q}_{11} \left(\frac{\partial u_0}{\partial x} + \frac{1}{2} \left(\frac{\partial w_0}{\partial x} \right)^2 - z \frac{\partial^2 w_0}{\partial x^2} \right) + \bar{Q}_{12} \left(\frac{\partial v_0}{\partial y} + \frac{1}{2} \left(\frac{\partial w_0}{\partial y} \right)^2 - z \frac{\partial^2 w_0}{\partial y^2} \right) \right\} dz \\
 &\quad + \int_{-h/2}^{h/2} \bar{Q}_{16} \left(\frac{\partial u_0}{\partial x} + \frac{\partial w_0}{\partial x} \frac{\partial w_0}{\partial y} - 2z \frac{\partial^2 w_0}{\partial x \partial y} \right) dz \\
 &= A_{11} \left[\frac{\partial u_0}{\partial x} + \frac{1}{2} \left(\frac{\partial w_0}{\partial x} \right)^2 \right] + A_{12} \left[\frac{\partial v_0}{\partial y} + \frac{1}{2} \left(\frac{\partial w_0}{\partial y} \right)^2 \right] + A_{16} \left[\frac{\partial u_0}{\partial y} + \frac{\partial v_0}{\partial x} + \frac{\partial w_0}{\partial x} \frac{\partial w_0}{\partial y} \right] \\
 &\quad - B_{11} \frac{\partial^2 w_0}{\partial x^2} - B_{12} \frac{\partial^2 w_0}{\partial y^2} - 2B_{16} \frac{\partial^2 w_0}{\partial x \partial y} \\
 M_{xx} &= \int_{-h/2}^{h/2} \sigma_{xx} z dz = \int_{-h/2}^{h/2} z \left[\bar{Q}_{11} \left(\frac{\partial u_0}{\partial x} + \frac{1}{2} \left(\frac{\partial w_0}{\partial x} \right)^2 - z \frac{\partial^2 w_0}{\partial x^2} \right) + \bar{Q}_{12} \left(\frac{\partial v_0}{\partial y} + \frac{1}{2} \left(\frac{\partial w_0}{\partial y} \right)^2 - z \frac{\partial^2 w_0}{\partial y^2} \right) \right] dz \\
 &\quad + \int_{-h/2}^{h/2} z \bar{Q}_{16} \left(\frac{\partial u_0}{\partial x} + \frac{\partial w_0}{\partial x} \frac{\partial w_0}{\partial y} - 2z \frac{\partial^2 w_0}{\partial x \partial y} \right) dz \\
 &= B_{11} \left[\frac{\partial u_0}{\partial x} + \frac{1}{2} \left(\frac{\partial w_0}{\partial x} \right)^2 \right] + B_{12} \left[\frac{\partial v_0}{\partial y} + \frac{1}{2} \left(\frac{\partial w_0}{\partial y} \right)^2 \right] + B_{16} \left[\frac{\partial u_0}{\partial y} + \frac{\partial v_0}{\partial x} + \frac{\partial w_0}{\partial x} \frac{\partial w_0}{\partial y} \right] \\
 &\quad - D_{11} \frac{\partial^2 w_0}{\partial x^2} - D_{12} \frac{\partial^2 w_0}{\partial y^2} - 2D_{16} \frac{\partial^2 w_0}{\partial x \partial y}
 \end{aligned}$$

Laminated Plate Constitutive Equations (Linear CLPT)

$$N_{xx} = A_{11} \frac{\partial u}{\partial x} + A_{12} \frac{\partial v}{\partial y} + A_{16} \left(\frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \right) - B_{11} \frac{\partial^2 w}{\partial x^2} - B_{12} \frac{\partial^2 w}{\partial y^2} - 2B_{16} \frac{\partial^2 w}{\partial x \partial y} - N_{xx}^T$$

$$N_{yy} = A_{12} \frac{\partial u}{\partial x} + A_{22} \frac{\partial v}{\partial y} + A_{26} \left(\frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \right) - B_{12} \frac{\partial^2 w}{\partial x^2} - B_{22} \frac{\partial^2 w}{\partial y^2} - 2B_{26} \frac{\partial^2 w}{\partial x \partial y} - N_{yy}^T$$

$$N_{xy} = A_{16} \frac{\partial u}{\partial x} + A_{26} \frac{\partial v}{\partial y} + A_{66} \left(\frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \right) - B_{16} \frac{\partial^2 w}{\partial x^2} - B_{26} \frac{\partial^2 w}{\partial y^2} - 2B_{66} \frac{\partial^2 w}{\partial x \partial y} - N_{xy}^T$$

$$M_{xx} = B_{11} \frac{\partial u}{\partial x} + B_{12} \frac{\partial v}{\partial y} + B_{16} \left(\frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \right) - D_{11} \frac{\partial^2 w}{\partial x^2} - D_{12} \frac{\partial^2 w}{\partial y^2} - 2D_{16} \frac{\partial^2 w}{\partial x \partial y} - M_{xx}^T$$

$$M_{yy} = B_{12} \frac{\partial u}{\partial x} + B_{22} \frac{\partial v}{\partial y} + B_{26} \left(\frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \right) - D_{12} \frac{\partial^2 w}{\partial x^2} - D_{22} \frac{\partial^2 w}{\partial y^2} - 2D_{26} \frac{\partial^2 w}{\partial x \partial y} - M_{yy}^T$$

$$M_{xy} = B_{16} \frac{\partial u}{\partial x} + B_{26} \frac{\partial v}{\partial y} + B_{66} \left(\frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \right) - D_{16} \frac{\partial^2 w}{\partial x^2} - D_{26} \frac{\partial^2 w}{\partial y^2} - 2D_{66} \frac{\partial^2 w}{\partial x \partial y} - M_{xy}^T$$

$$Q_x = K_s A_{55} \left(\phi_x + \frac{\partial w}{\partial x} \right) + K_s A_{45} \left(\phi_y + \frac{\partial w}{\partial y} \right); \quad Q_y = K_s A_{45} \left(\phi_x + \frac{\partial w}{\partial x} \right) + K_s A_{44} \left(\phi_y + \frac{\partial w}{\partial y} \right)$$

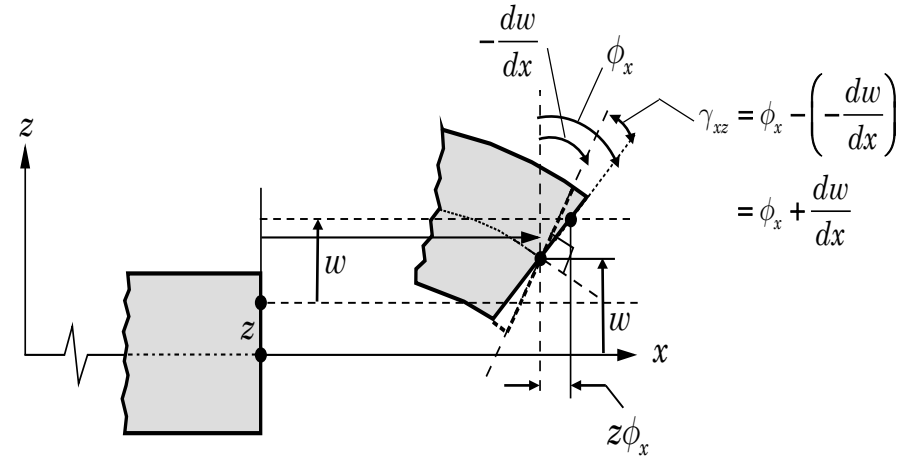
THE FIRST-ORDER SHEAR DEFORMATION THEORY (FSDT)

Displacement Field

$$u_1(x, y, z, t) = u(x, y, t) + z\phi_x(x, y, t)$$

$$u_2(x, y, z, t) = v(x, y, t) + z\phi_y(x, y, t)$$

$$u_3(x, y, z, t) = w(x, y, t)$$



Linear strains

$$\varepsilon_{xx} = \frac{\partial u}{\partial x} + z \frac{\partial \phi_x}{\partial x}, \quad \varepsilon_{yy} = \frac{\partial v}{\partial y} + z \frac{\partial \phi_y}{\partial x}, \quad \gamma_{xy} = \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} + z \left(\frac{\partial \phi_x}{\partial y} + \frac{\partial \phi_y}{\partial x} \right)$$

$$\gamma_{xz} = \left(\phi_x + \frac{\partial w}{\partial x} \right), \quad \gamma_{yz} = \left(\phi_y + \frac{\partial w}{\partial y} \right)$$

EQUATIONS OF MOTION OF FSDT

Equations of motion (FSDT)

$$\begin{array}{l}
 \sum F_x = 0 : \frac{\partial N_{xx}}{\partial x} + \frac{\partial N_{xy}}{\partial y} + f_x = I_0 \frac{\partial^2 u}{\partial t^2} \\
 \sum F_y = 0 : \frac{\partial N_{xy}}{\partial x} + \frac{\partial N_{yy}}{\partial y} + f_y = I_0 \frac{\partial^2 v}{\partial t^2} \\
 \sum F_z = 0 : \frac{\partial Q_x}{\partial x} + \frac{\partial Q_y}{\partial y} + q = I_0 \frac{\partial^2 w}{\partial t^2}
 \end{array} \left| \begin{array}{l}
 \sum M_y = 0 : \frac{\partial M_{xx}}{\partial x} + \frac{\partial M_{xy}}{\partial y} - Q_x = I_2 \frac{\partial^2 \phi_x}{\partial t^2} \\
 \sum M_x = 0 : \frac{\partial M_{xy}}{\partial x} + \frac{\partial M_{yy}}{\partial y} - Q_y = I_2 \frac{\partial^2 \phi_y}{\partial t^2}
 \end{array} \right.$$

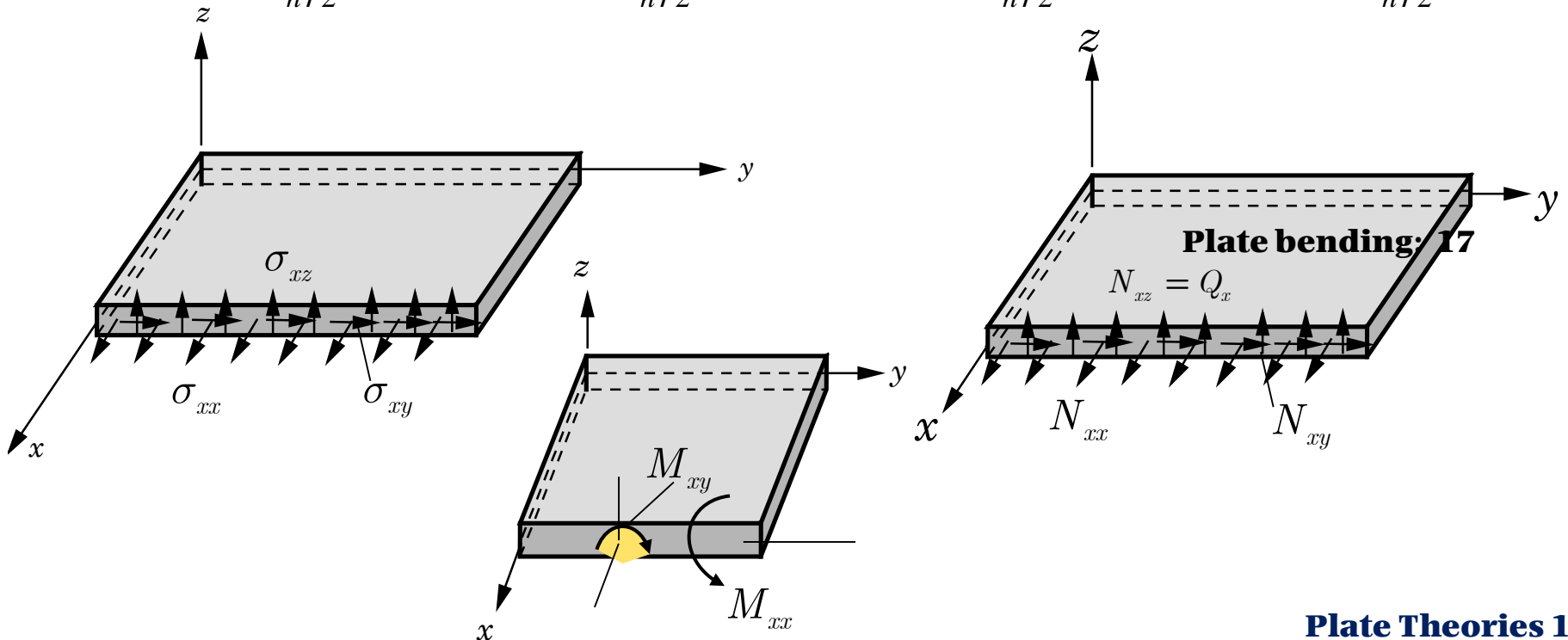
Definitions of stress resultants

$$\begin{array}{l}
 N_{xx} = \int_{-h/2}^{h/2} \sigma_{xx} dz, \quad N_{yy} = \int_{-h/2}^{h/2} \sigma_{yy} dz, \quad N_{xy} = \int_{-h/2}^{h/2} \sigma_{xy} dz, \quad Q_x = K_s \int_{-h/2}^{h/2} \sigma_{xz} dz \\
 M_{xx} = \int_{-h/2}^{h/2} z \sigma_{xx} dz, \quad M_{yy} = \int_{-h/2}^{h/2} z \sigma_{yy} dz, \quad M_{xy} = \int_{-h/2}^{h/2} z \sigma_{xy} dz, \quad Q_y = K_s \int_{-h/2}^{h/2} \sigma_{yz} dz
 \end{array}$$

Stresses and Stress Resultants on an edge of a Plate

$$N_{xx} = \int_{-h/2}^{h/2} \sigma_{xx} dz, \quad N_{yy} = \int_{-h/2}^{h/2} \sigma_{yy} dz, \quad N_{xy} = \int_{-h/2}^{h/2} \sigma_{xy} dz, \quad Q_x = K_s \int_{-h/2}^{h/2} \sigma_{xz} dz$$

$$M_{xx} = \int_{-h/2}^{h/2} z \sigma_{xx} dz, \quad M_{yy} = \int_{-h/2}^{h/2} z \sigma_{yy} dz, \quad M_{xy} = \int_{-h/2}^{h/2} z \sigma_{xy} dz, \quad Q_y = K_s \int_{-h/2}^{h/2} \sigma_{yz} dz$$



Navier Solution of Simply Supported Orthotropic Plates (CLPT)

$$- \left[D_{11} \frac{\partial^4 w_0}{\partial x^4} + 2(D_{12} + 2D_{66}) \frac{\partial^4 w_0}{\partial x^2 \partial y^2} + D_{22} \frac{\partial^4 w_0}{\partial y^4} \right] + q = 0$$

$$w_0(x, y) = \sum_{n=1}^{\infty} \sum_{m=1}^{\infty} W_{mn} \sin \alpha x \sin \beta y$$

$$q(x, y) = \sum_{n=1}^{\infty} \sum_{m=1}^{\infty} Q_{mn} \sin \alpha x \sin \beta y$$

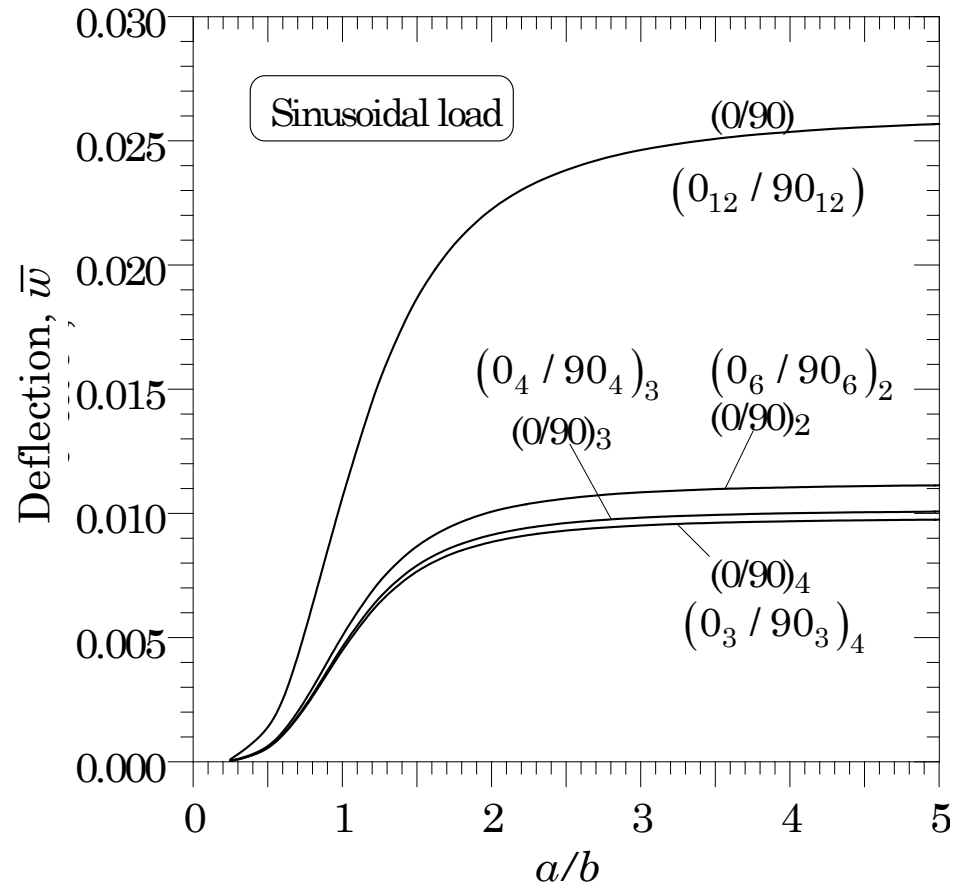
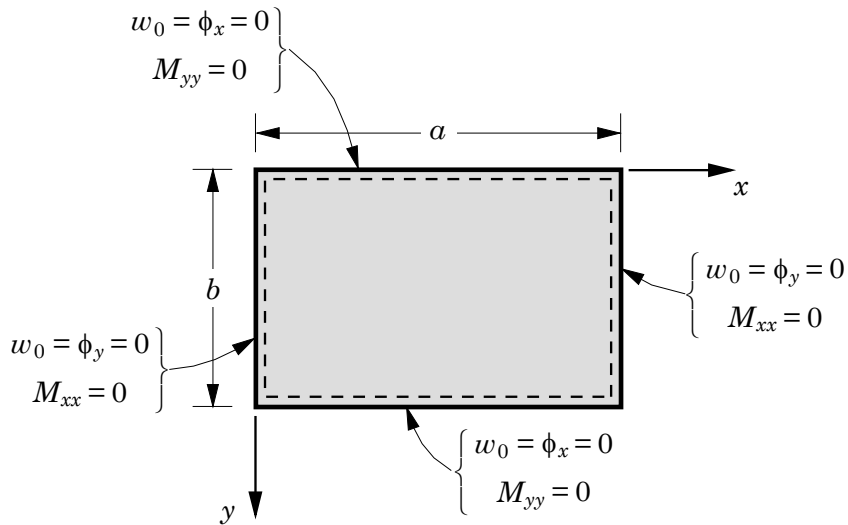
$$Q_{mn} = \frac{4}{ab} \int_0^b \int_0^a q(x, y) \sin \alpha x \sin \beta y \, dx dy$$

$$\sum_{n=1}^{\infty} \sum_{m=1}^{\infty} \left\{ -W_{mn} [D_{11}\alpha^4 + 2(D_{12} + 2D_{66})\alpha^2\beta^2 + D_{22}\beta^4] + Q_{mn} \right\} \sin \alpha x \sin \beta y = 0$$

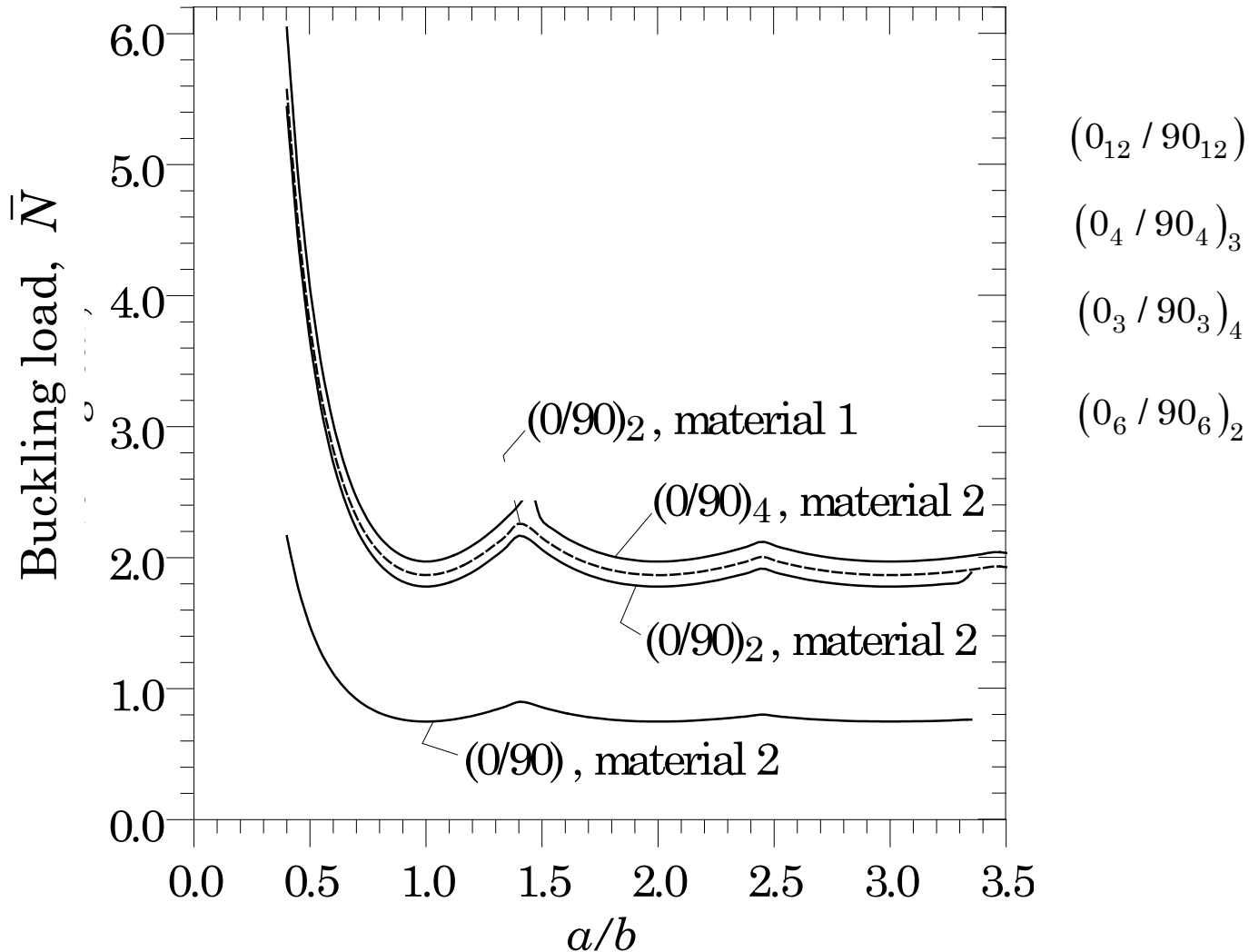
$$W_{mn} = \frac{Q_{mn}}{d_{mn}}, \quad d_{mn} = \frac{\pi^4}{b^4} [D_{11}m^4s^4 + 2(D_{12} + 2D_{66})m^2n^2s^2 + D_{22}n^4]$$

ANALYTICAL SOLUTIONS

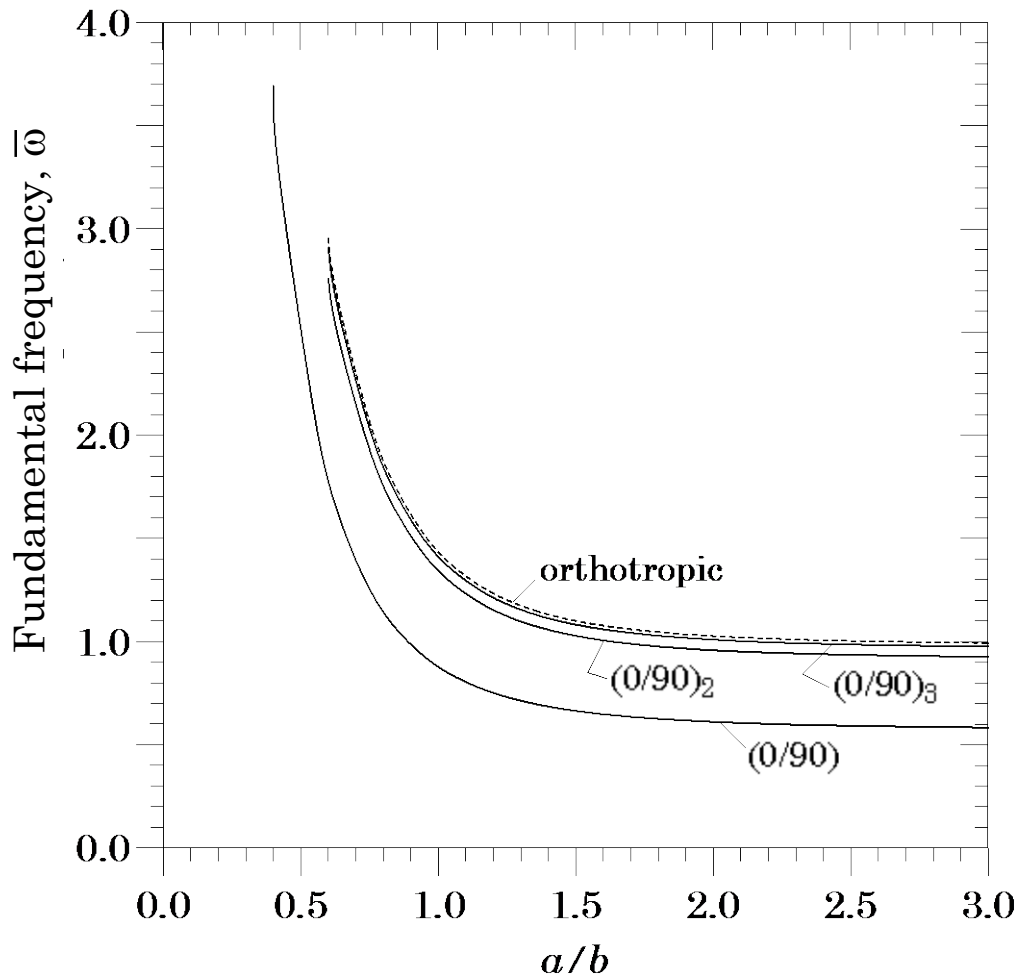
Bending of Simply Supported Plates (SS-1)-CLPT



Buckling of Plates (SS-1) - CLPT



Vibration of Plates (SS-1) - CLPT



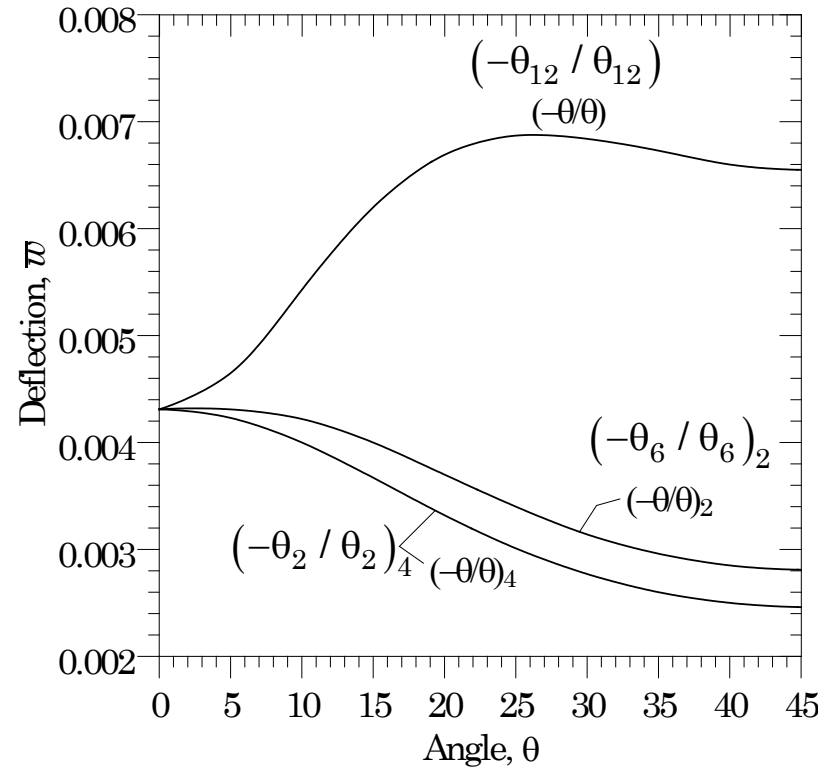
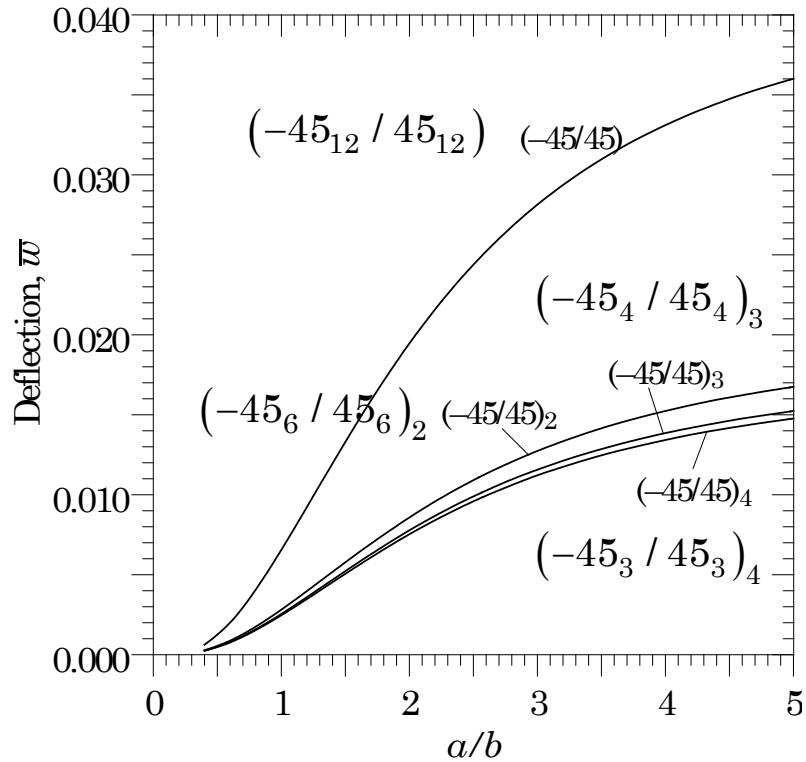
$$(0/90) = (0_{12}/90_{12})$$

$$(0/90)_2 = (0_6/90_6)_2$$

$$(0/90)_4 = (0_3/90_3)_4$$

$$(0/90)_3 = (0_4/90_4)_3$$

Bending of Angle-ply Plates (SS-2)-CLPT



BUCKLING OF LAMINATED COMPOSITE PLATES USING FSDT

$$\begin{aligned} \frac{\partial}{\partial x} \left[A_{11} \frac{\partial u_0}{\partial x} + A_{12} \frac{\partial v_0}{\partial y} + A_{16} \left(\frac{\partial u_0}{\partial y} + \frac{\partial v_0}{\partial x} \right) + B_{11} \frac{\partial \phi_x}{\partial x} + B_{12} \frac{\partial \phi_y}{\partial y} \right. \\ \left. + B_{16} \left(\frac{\partial \phi_x}{\partial y} + \frac{\partial \phi_y}{\partial x} \right) \right] + \frac{\partial}{\partial y} \left[A_{16} \frac{\partial u_0}{\partial x} + A_{26} \frac{\partial v_0}{\partial y} \right. \\ \left. + A_{66} \left(\frac{\partial u_0}{\partial y} + \frac{\partial v_0}{\partial x} \right) + B_{16} \frac{\partial \phi_x}{\partial x} + B_{26} \frac{\partial \phi_y}{\partial y} + B_{66} \left(\frac{\partial \phi_x}{\partial y} + \frac{\partial \phi_y}{\partial x} \right) \right] = 0 \end{aligned}$$

$$\begin{aligned} \frac{\partial}{\partial x} \left[A_{16} \frac{\partial u_0}{\partial x} + A_{26} \frac{\partial v_0}{\partial y} + A_{66} \left(\frac{\partial u_0}{\partial y} + \frac{\partial v_0}{\partial x} \right) + B_{16} \frac{\partial \phi_x}{\partial x} + B_{26} \frac{\partial \phi_y}{\partial y} \right. \\ \left. + B_{66} \left(\frac{\partial \phi_x}{\partial y} + \frac{\partial \phi_y}{\partial x} \right) \right] + \frac{\partial}{\partial y} \left[A_{12} \frac{\partial u_0}{\partial x} + A_{22} \frac{\partial v_0}{\partial y} \right. \\ \left. + A_{26} \left(\frac{\partial u_0}{\partial y} + \frac{\partial v_0}{\partial x} \right) + B_{12} \frac{\partial \phi_x}{\partial x} + B_{22} \frac{\partial \phi_y}{\partial y} + B_{26} \left(\frac{\partial \phi_x}{\partial y} + \frac{\partial \phi_y}{\partial x} \right) \right] = 0 \end{aligned}$$

$$\begin{aligned} \frac{\partial}{\partial x} \left[K A_{45} \left(\frac{\partial w_0}{\partial y} + \phi_y \right) + K A_{55} \left(\frac{\partial w_0}{\partial x} + \phi_x \right) \right] \\ + \frac{\partial}{\partial y} \left[K A_{44} \left(\frac{\partial w_0}{\partial y} + \phi_y \right) + K A_{45} \left(\frac{\partial w_0}{\partial x} + \phi_x \right) \right] \\ + \hat{N}_{xx} \frac{\partial^2 w_0}{\partial x^2} + 2\hat{N}_{xy} \frac{\partial^2 w_0}{\partial x \partial y} + \hat{N}_{yy} \frac{\partial^2 w_0}{\partial y^2} = 0 \end{aligned}$$

BUCKLING OF LAMINATED COMPOSITE PLATES USING FSDT

Governing Equations (continued)

$$\begin{aligned} \frac{\partial}{\partial x} \left[B_{11} \frac{\partial u_0}{\partial x} + B_{12} \frac{\partial v_0}{\partial y} + B_{16} \left(\frac{\partial u_0}{\partial y} + \frac{\partial v_0}{\partial x} \right) + D_{11} \frac{\partial \phi_x}{\partial x} + D_{12} \frac{\partial \phi_y}{\partial y} \right. \\ \left. + D_{16} \left(\frac{\partial \phi_x}{\partial y} + \frac{\partial \phi_y}{\partial x} \right) \right] + \frac{\partial}{\partial y} \left[B_{16} \frac{\partial u_0}{\partial x} + B_{26} \frac{\partial v_0}{\partial y} \right. \\ \left. + B_{66} \left(\frac{\partial u_0}{\partial y} + \frac{\partial v_0}{\partial x} \right) + D_{16} \frac{\partial \phi_x}{\partial x} + D_{26} \frac{\partial \phi_y}{\partial y} + D_{66} \left(\frac{\partial \phi_x}{\partial y} + \frac{\partial \phi_y}{\partial x} \right) \right] \\ - \left[K A_{45} \left(\frac{\partial w_0}{\partial y} + \phi_y \right) + K A_{55} \left(\frac{\partial w_0}{\partial x} + \phi_x \right) \right] = 0 \end{aligned}$$

$$\begin{aligned} \frac{\partial}{\partial x} \left[B_{16} \frac{\partial u_0}{\partial x} + B_{26} \frac{\partial v_0}{\partial y} + B_{66} \left(\frac{\partial u_0}{\partial y} + \frac{\partial v_0}{\partial x} \right) + D_{16} \frac{\partial \phi_x}{\partial x} \right. \\ \left. + D_{26} \frac{\partial \phi_y}{\partial y} + D_{66} \left(\frac{\partial \phi_x}{\partial y} + \frac{\partial \phi_y}{\partial x} \right) \right] + \frac{\partial}{\partial y} \left[B_{12} \frac{\partial u_0}{\partial x} + B_{22} \frac{\partial v_0}{\partial y} \right. \\ \left. + B_{26} \left(\frac{\partial u_0}{\partial y} + \frac{\partial v_0}{\partial x} \right) + D_{12} \frac{\partial \phi_x}{\partial y} + D_{22} \frac{\partial \phi_y}{\partial y} + D_{26} \left(\frac{\partial \phi_x}{\partial y} + \frac{\partial \phi_y}{\partial x} \right) \right] \\ - \left[K A_{44} \left(\frac{\partial w_0}{\partial y} + \phi_y \right) + K A_{45} \left(\frac{\partial w_0}{\partial x} + \phi_x \right) \right] = 0 \end{aligned}$$

BUCKLING OF LAMINATED COMPOSITE PLATES USING FSDT

The Navier Solution

SS-1 Boundary Conditions

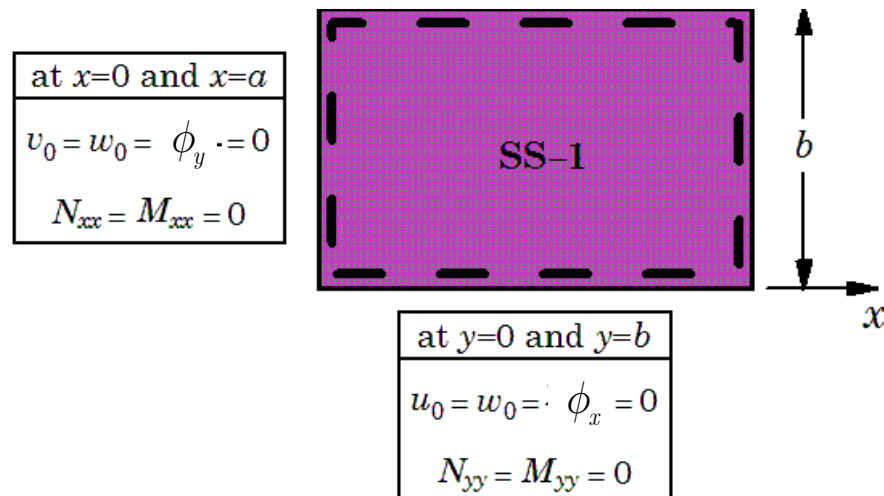
$$u_0(x, 0) = 0, \quad u_0(x, b) = 0, \quad v_0(0, y) = 0, \quad v_0(a, y) = 0$$

$$w_0(x, 0) = 0, \quad w_0(x, b) = 0, \quad w_0(0, y) = 0, \quad w_0(a, y) = 0$$

$$\phi_x(x, 0) = 0, \quad \phi_x(x, b) = 0, \quad \phi_y(0, y) = 0, \quad \phi_y(a, y) = 0$$

$$N_{xx}(0, y) = 0, \quad N_{xx}(a, y) = 0, \quad N_{yy}(x, 0) = 0, \quad N_{yy}(x, b) = 0$$

$$M_{xx}(0, y) = 0, \quad M_{xx}(a, y) = 0, \quad M_{yy}(x, 0) = 0, \quad M_{yy}(x, b) = 0$$



BUCKLING OF *ANTISYMMETRIC CROSS-PLY* LAMINATES USING FSDT

The Navier Solution

$$u_0(x, y, t) = \sum_{n=1}^{\infty} \sum_{m=1}^{\infty} U_{mn} \cos \alpha x \sin \beta y$$

$$v_0(x, y, t) = \sum_{n=1}^{\infty} \sum_{m=1}^{\infty} V_{mn} \sin \alpha x \cos \beta y$$

$$w_0(x, y, t) = \sum_{n=1}^{\infty} \sum_{m=1}^{\infty} W_{mn} \sin \alpha x \sin \beta y$$

$$\phi_x(x, y, t) = \sum_{n=1}^{\infty} \sum_{m=1}^{\infty} X_{mn} \cos \alpha x \sin \beta y$$

$$\phi_y(x, y, t) = \sum_{n=1}^{\infty} \sum_{m=1}^{\infty} Y_{mn} \sin \alpha x \cos \beta y$$

where $\alpha = m\pi/a$ and $\beta = n\pi/b$.

$$A_{16} = 0, \quad A_{26} = 0, \quad A_{45} = 0, \quad B_{16} = 0, \quad B_{26} = 0, \quad D_{16} = 0, \quad D_{26} = 0$$

$$\hat{N}_{xx} = -N_0, \quad \hat{N}_{yy} = -kN_0, \quad k = \frac{\hat{N}_{yy}}{\hat{N}_{xx}}$$

BUCKLING OF *ANTISYMMETRIC CROSS-PLY* LAMINATES USING FSDT

The Navier Solution

$$\begin{bmatrix} s_{11} & s_{12} & 0 & s_{14} & s_{15} \\ s_{12} & s_{22} & 0 & s_{24} & s_{25} \\ 0 & 0 & s_{33} - N_0(\alpha^2 + k\beta^2) & s_{34} & s_{35} \\ s_{14} & s_{24} & s_{34} & s_{44} & s_{45} \\ s_{15} & s_{25} & s_{35} & s_{45} & s_{55} \end{bmatrix} \begin{Bmatrix} U_{mn} \\ V_{mn} \\ W_{mn} \\ X_{mn} \\ Y_{mn} \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \\ 0 \\ 0 \\ 0 \end{Bmatrix}$$

$$s_{11} = (A_{11}\alpha^2 + A_{66}\beta^2), \quad s_{12} = (A_{12} + A_{66})\alpha\beta$$

$$s_{14} = (B_{11}\alpha^2 + B_{66}\beta^2), \quad s_{15} = (B_{12} + B_{66})\alpha\beta$$

$$s_{22} = (A_{66}\alpha^2 + A_{22}\beta^2), \quad s_{24} = s_{15}$$

$$s_{25} = (B_{66}\alpha^2 + B_{22}\beta^2), \quad s_{33} = K(A_{55}\alpha^2 + A_{44}\beta^2)$$

$$s_{34} = KA_{55}\alpha, \quad s_{35} = KA_{44}\beta, \quad s_{44} = (D_{11}\alpha^2 + D_{66}\beta^2 + KA_{55})$$

$$s_{45} = (D_{12} + D_{66})\alpha\beta, \quad s_{55} = (D_{66}\alpha^2 + D_{22}\beta^2 + KA_{44})$$

BUCKLING OF *ANTISYMMETRIC CROSS-PLY* LAMINATES USING FSDT

The Navier Solution

$$N_0 = \left(\frac{1}{\alpha^2 + k\beta^2} \right) \left(\frac{K^2 A_{44} A_{55} c_{33} + (K A_{55} \alpha^2 + K A_{44} \beta^2) c_1}{c_1 + K A_{44} c_2 + K A_{55} c_3 + K^2 A_{44} A_{55}} \right)$$

$$= \left(\frac{1}{\alpha^2 + k\beta^2} \right) \left(\frac{c_{33} + \left(\frac{\alpha^2}{K A_{44}} + \frac{\beta^2}{K A_{55}} \right) c_1}{1 + \frac{c_1}{K^2 A_{44} A_{55}} + \frac{c_2}{K A_{55}} + \frac{c_3}{K A_{44}}} \right)$$

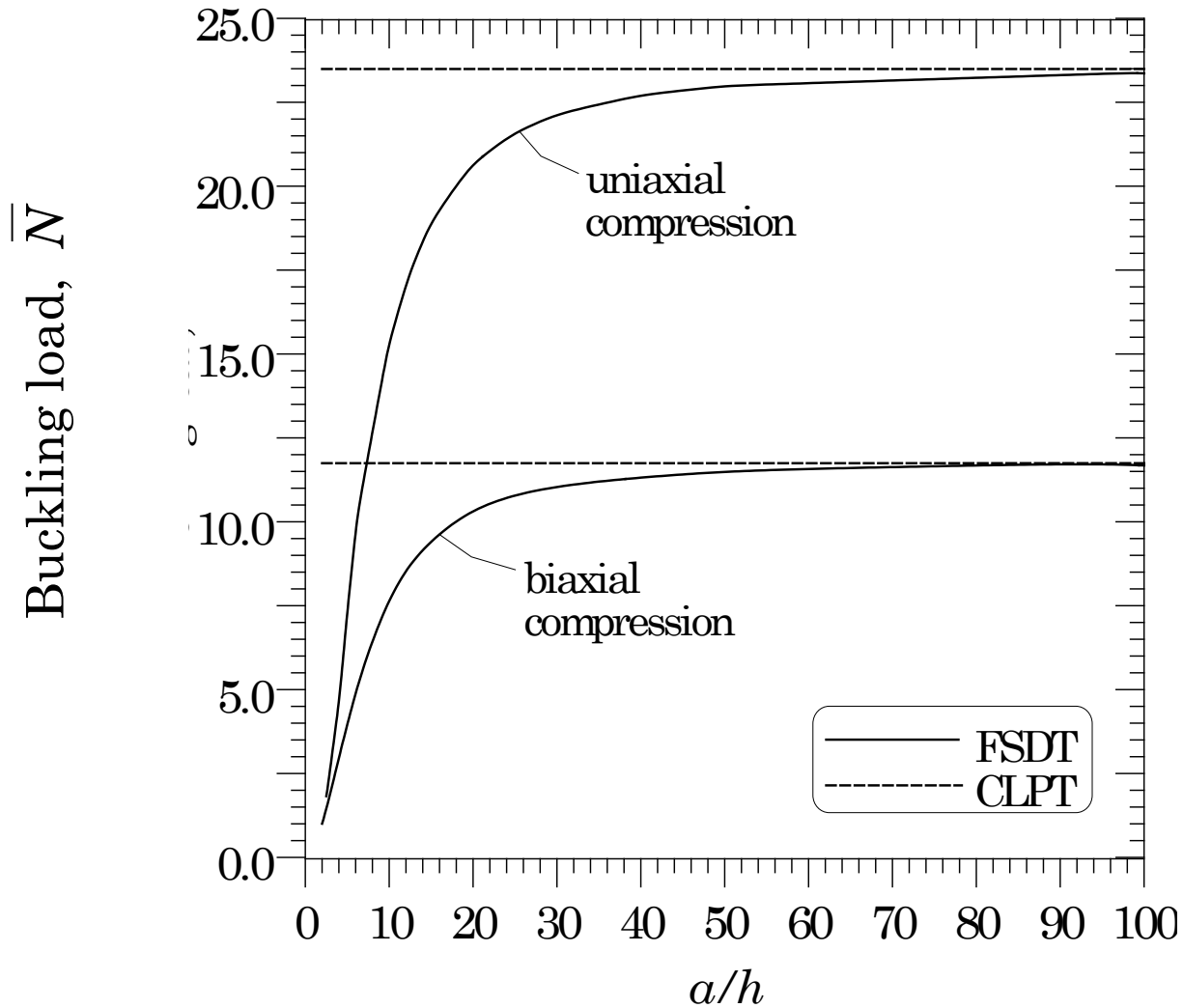
$$c_{33} = D_{11} \alpha^4 + 2(D_{12} + 2D_{66}) \alpha^2 \beta^2 + D_{22} \beta^4, \quad c_1 = c_2 c_3 - (c_4)^2 > 0$$

$$c_2 = D_{11} \alpha^2 + D_{66} \beta^2, \quad c_3 = D_{66} \alpha^2 + D_{22} \beta^2, \quad c_4 = (D_{12} + D_{66}) \alpha \beta$$

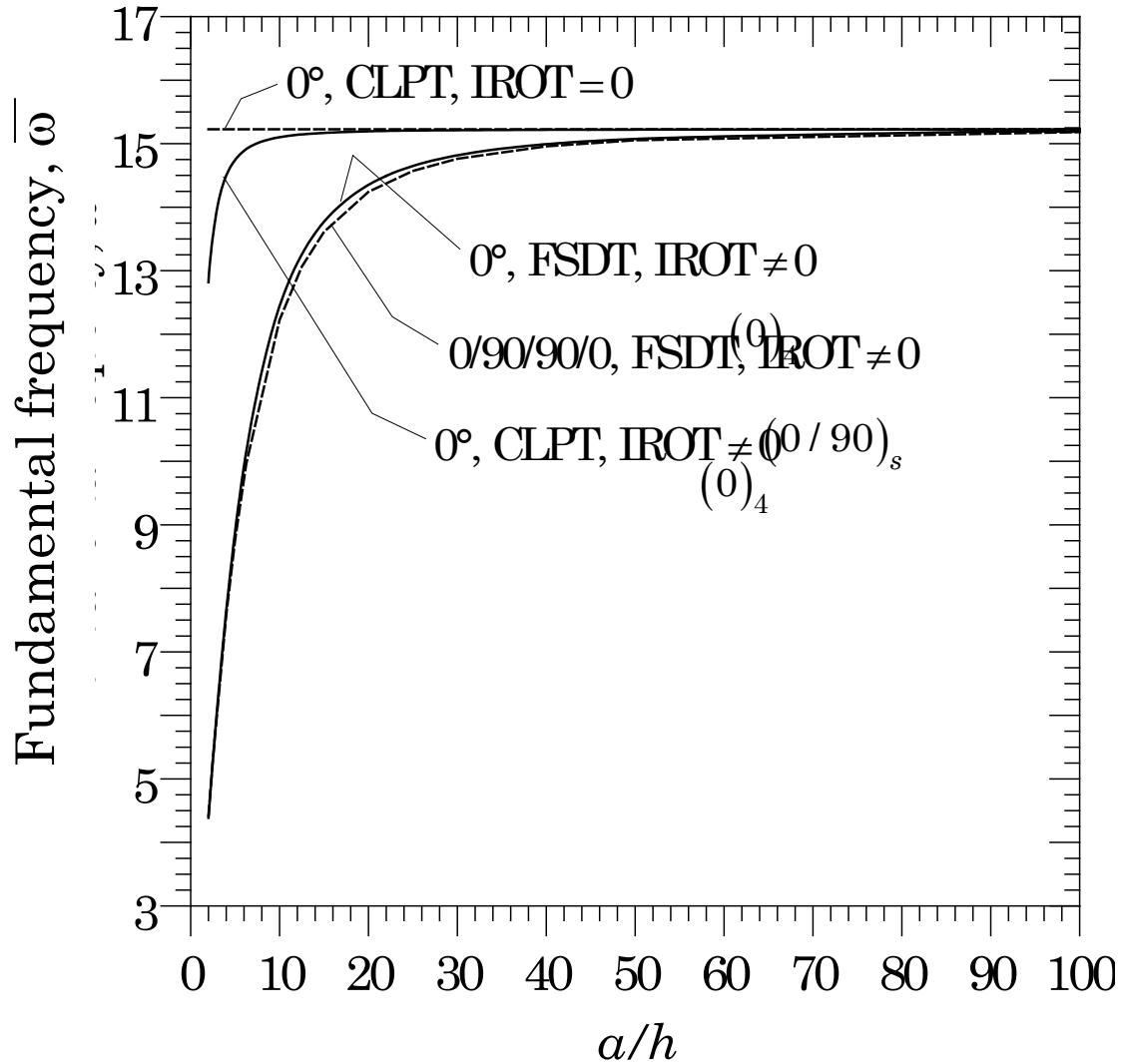
$$\frac{c_{33} + k_1}{1 + k_2} \quad \text{where} \quad k_1 < k_2, \quad c_{33} \geq \frac{c_{33} + k_1}{1 + k_2}$$

$$N_{cr} = 4D \left(\frac{\pi}{a} \right)^2 \frac{\left[1 + \frac{3(1-\nu^2)\pi^2(h/a)^2}{K} \right]}{\left[1 + \frac{72(1+\nu)(1-\nu^2)\pi^4(h/a)^4}{K^2} + \frac{6(1+\nu)(3-\nu)\pi^2(h/a)^2}{K} \right]}$$

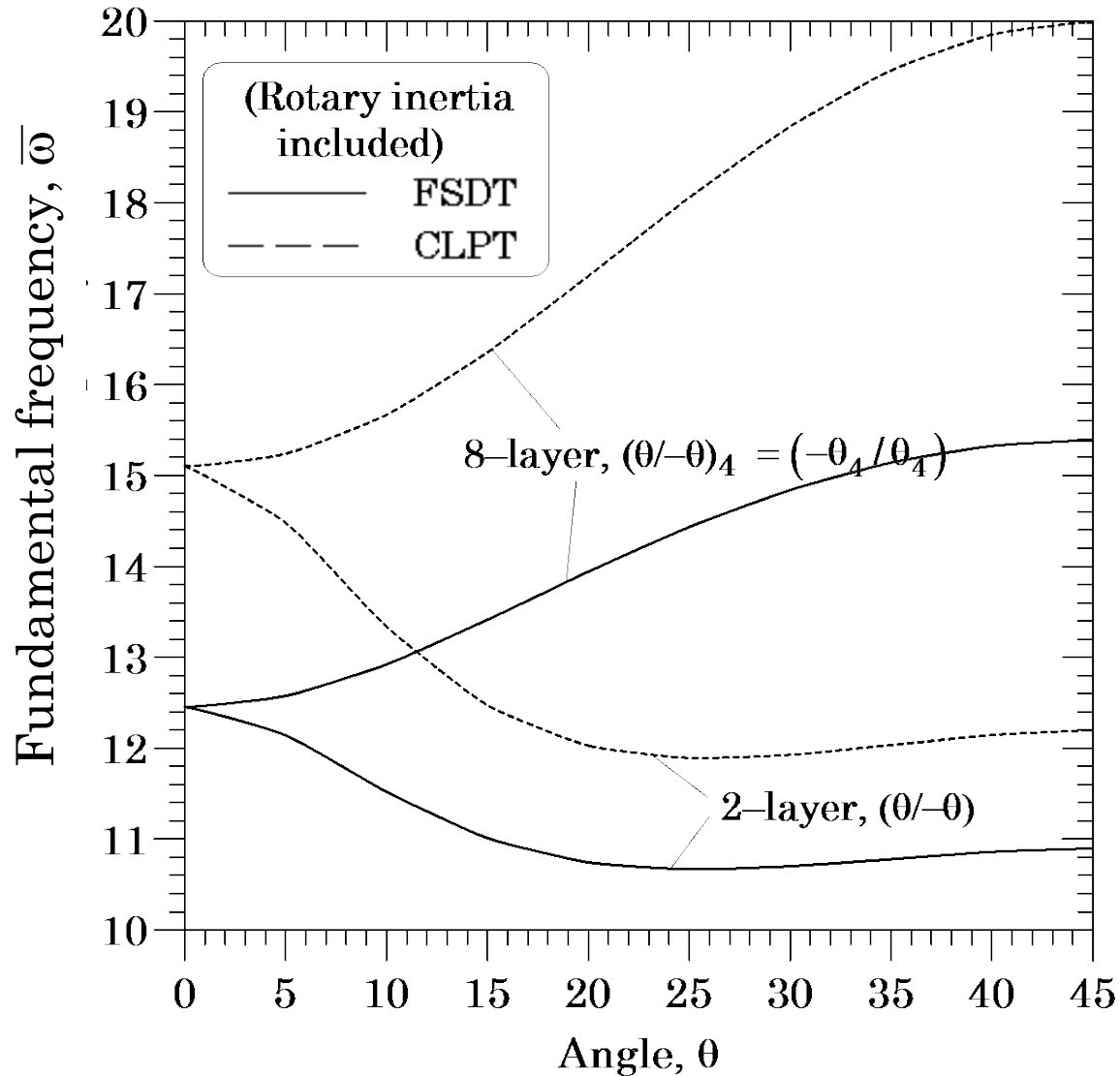
Effect of Shear Deformation on Buckling



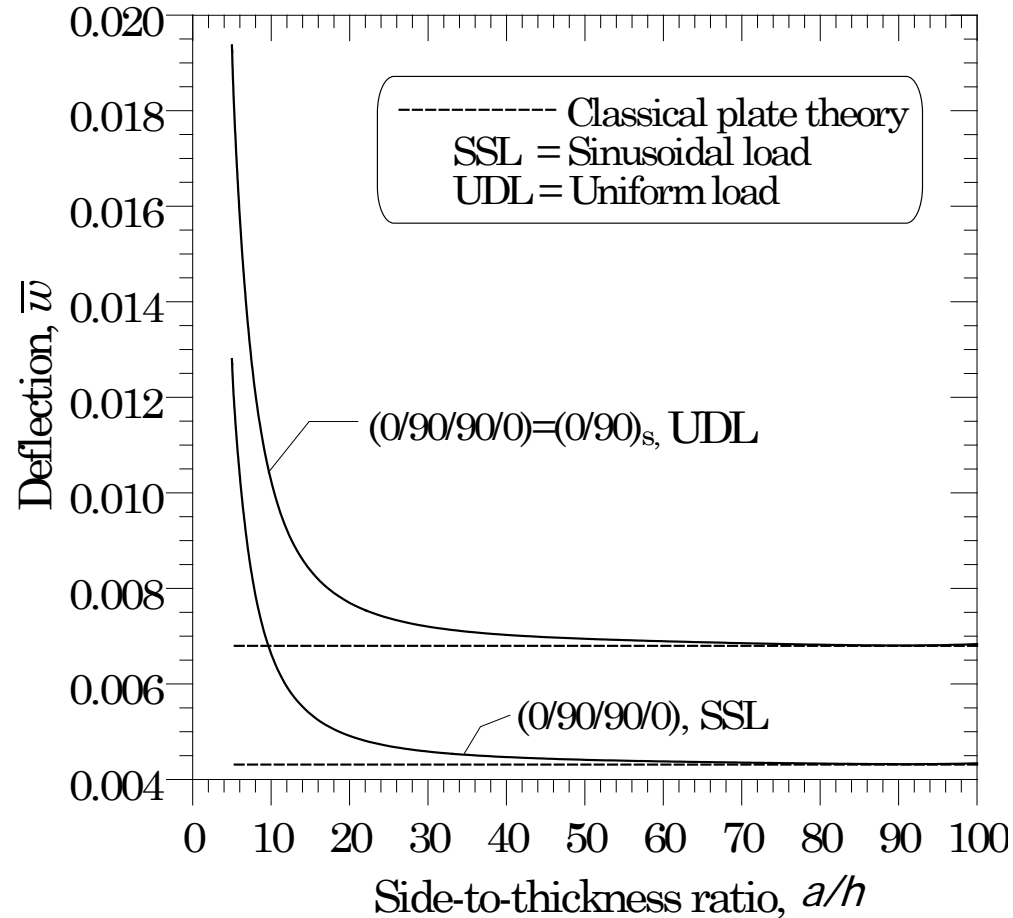
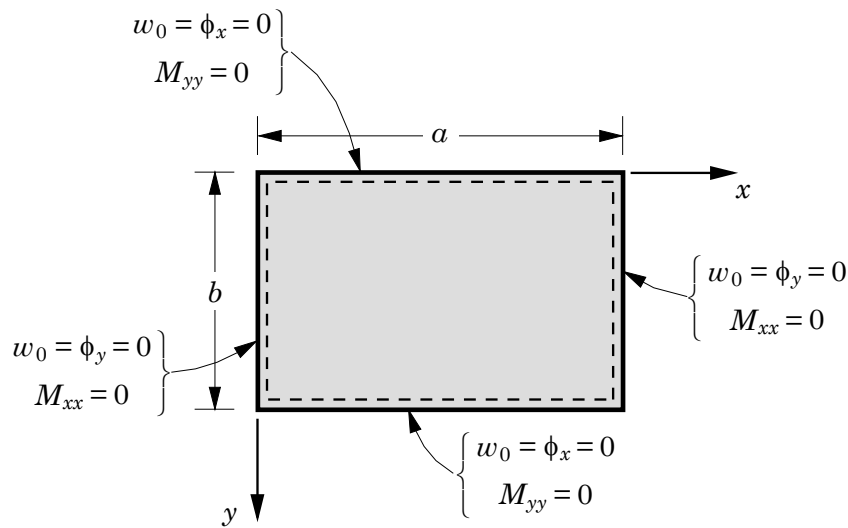
Effect of Shear Deformation On Vibration



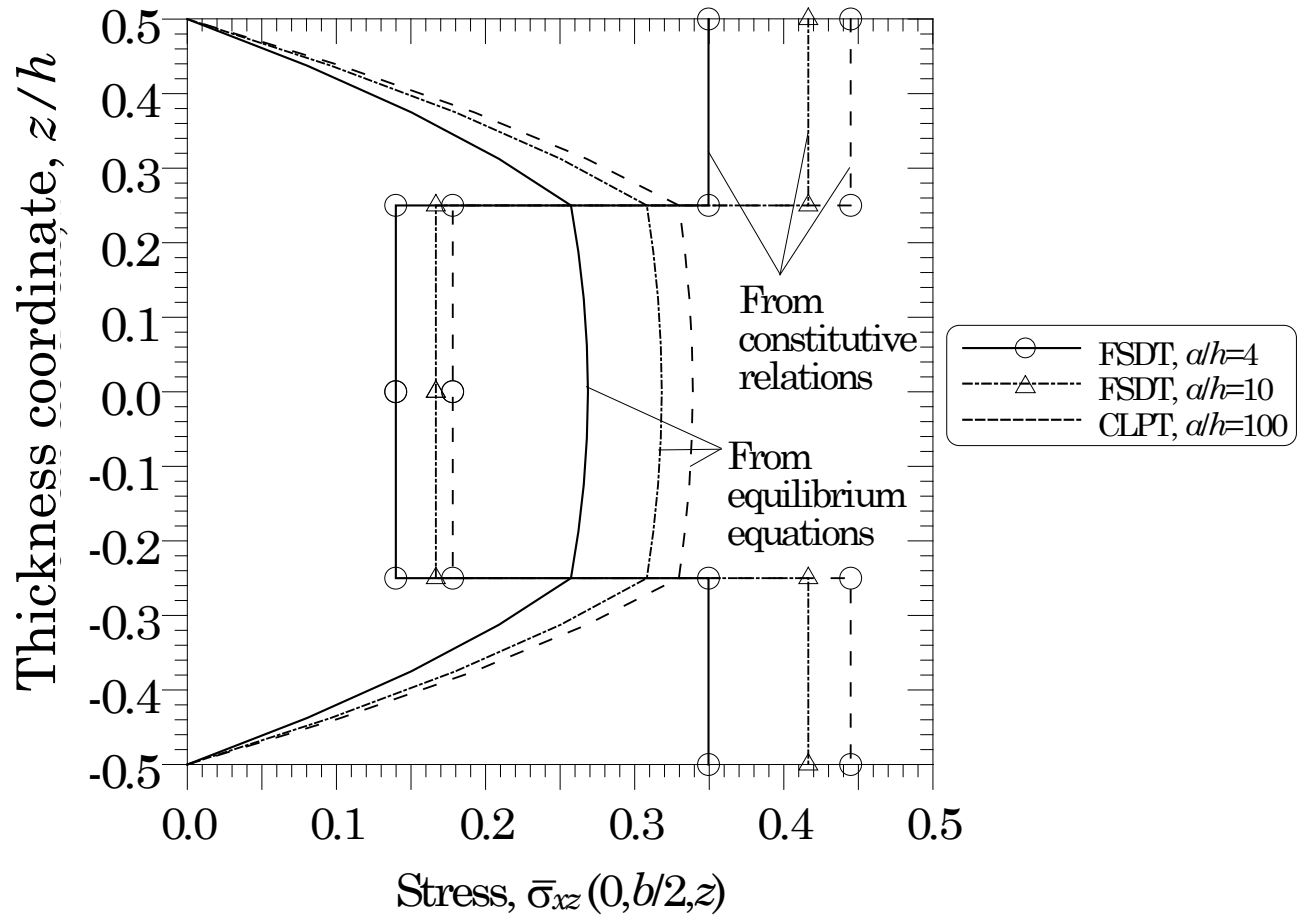
Shear Deformation in Angle-Ply Plates



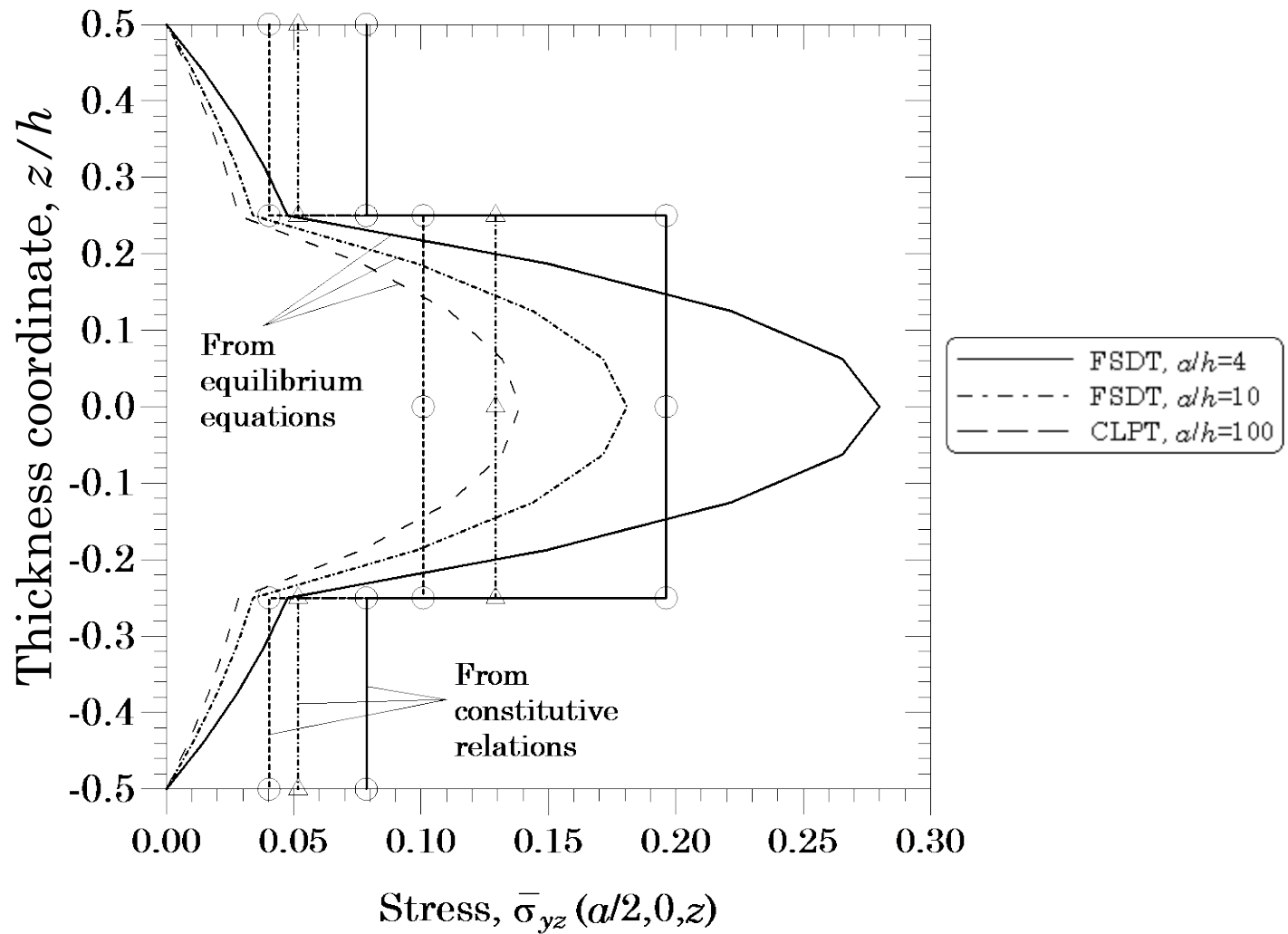
Effect of Shear Deformation on Bending Deformation



Transverse Shear Stresses



Transverse Shear Stresses



Shear Deformation in Angle-Ply Plates

