



HIGHER-ORDER THEORIES

- **Third-order Shear Deformation Plate Theory**
 - **Displacement and strain fields**
 - **Equations of motion**
 - **Navier's solution for bending**

- **Layerwise Laminate Theory**
 - **Interlaminar stress and strain continuity**
 - **Equations of motion**
 - **Numerical results**

TSDT–Displacement Field (continued)

Reduction of the Displacement Field : Require that the top and bottom faces of the plate are free of shear stress

$$\sigma_{xz}(x, y, \pm \frac{h}{2}, t) = \sigma_{yz}(x, y, \pm \frac{h}{2}, t) = 0$$

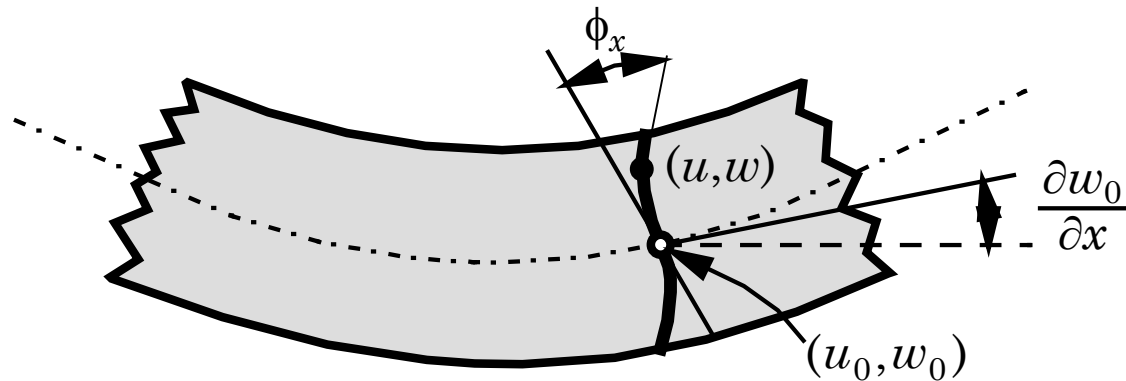
$$\Rightarrow \gamma_{xz}(x, y, \pm \frac{h}{2}, t) = \gamma_{yz}(x, y, \pm \frac{h}{2}, t) = 0$$

$$\gamma_{xz} = \phi_x + \frac{\partial w}{\partial x} + 2z\psi_x + 3z^2\theta_x$$

$$\phi_x + \frac{\partial w}{\partial x} - h\psi_x + \frac{3h^2}{4}\theta_x = 0, \quad \phi_x + \frac{\partial w}{\partial x} + h\psi_x + \frac{3h^2}{4}\theta_x = 0$$

$$\Rightarrow \theta_x = -\frac{4}{3h^2} \left(\phi_x + \frac{\partial w}{\partial x} \right), \quad \psi_x = 0$$

Third-Order Shear Deformation Plate Theory (TSDT) – Displacement Field



Assumed Displacement Field

$$u_1(x, y, z, t) = u(x, y, t) + z\phi_x(x, y, t) + z^2\psi_x(x, y, t) + z^3\theta_x(x, y, t)$$

$$u_2(x, y, z, t) = v(x, y, t) + z\phi_y(x, y, t) + z^2\psi_y(x, y, t) + z^3\theta_y(x, y, t)$$

$$u_3(x, y, z, t) = w(x, y, t)$$

Displacement Field of the Reddy Third-Order Laminate Plate Theory (RLPT)

Displacement Field

$$u_1(x, y, z, t) = u(x, y, t) + z\phi_x(x, y, t) + z^3 \left(-\frac{4}{3h^2} \right) \left(\phi_x + \frac{\partial w}{\partial x} \right)$$

$$u_2(x, y, z, t) = v(x, y, t) + z\phi_y(x, y, t) + z^3 \left(-\frac{4}{3h^2} \right) \left(\phi_y + \frac{\partial w}{\partial y} \right)$$

$$u_3(x, y, z, t) = w(x, y, t)$$

Strain Field

$$\begin{Bmatrix} \varepsilon_{xx} \\ \varepsilon_{yy} \\ 2\varepsilon_{xy} \end{Bmatrix} = \begin{Bmatrix} \varepsilon_{xx} \\ \varepsilon_{yy} \\ 2\varepsilon_{xy} \end{Bmatrix}^{(0)} + z \begin{Bmatrix} \varepsilon_{xx} \\ \varepsilon_{yy} \\ 2\varepsilon_{xy} \end{Bmatrix}^{(1)} + z^3 \begin{Bmatrix} \varepsilon_{xx} \\ \varepsilon_{yy} \\ 2\varepsilon_{xy} \end{Bmatrix}^{(3)}$$

$$\begin{Bmatrix} \gamma_{xz} \\ \gamma_{yz} \end{Bmatrix} = \begin{Bmatrix} \gamma_{xz} \\ \gamma_{yz} \end{Bmatrix}^{(0)} + z^2 \begin{Bmatrix} \gamma_{xz} \\ \gamma_{yz} \end{Bmatrix}^{(2)}$$

Strain Field of the Reddy Third-Order Laminate Plate Theory

Strain Field (continued)

$$\begin{aligned}
 \begin{Bmatrix} \varepsilon_{xx} \\ \varepsilon_{yy} \\ 2\varepsilon_{xy} \end{Bmatrix}^{(0)} &= \begin{Bmatrix} \frac{\partial u}{\partial x} + \frac{1}{2} \left(\frac{\partial w}{\partial x} \right)^2 \\ \frac{\partial v}{\partial y} + \frac{1}{2} \left(\frac{\partial w}{\partial y} \right)^2 \\ \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} + \frac{\partial w}{\partial x} \frac{\partial w}{\partial y} \end{Bmatrix}, \quad \begin{Bmatrix} \varepsilon_{xx} \\ \varepsilon_{yy} \\ 2\varepsilon_{xy} \end{Bmatrix}^{(1)} = \begin{Bmatrix} \frac{\partial \phi_x}{\partial x} \\ \frac{\partial \phi_y}{\partial y} \\ \frac{\partial \phi_x}{\partial y} + \frac{\partial \phi_y}{\partial x} \end{Bmatrix}, \quad c_1 = \frac{4}{3h^2} \\
 \begin{Bmatrix} \varepsilon_{xx} \\ \varepsilon_{yy} \\ 2\varepsilon_{xy} \end{Bmatrix}^{(3)} &= -c_1 \begin{Bmatrix} \frac{\partial \phi_x}{\partial x} + \frac{\partial^2 w}{\partial x^2} \\ \frac{\partial \phi_y}{\partial y} + \frac{\partial^2 w}{\partial y^2} \\ \frac{\partial \phi_x}{\partial y} + \frac{\partial \phi_y}{\partial x} + 2 \frac{\partial^2 w}{\partial x \partial y} \end{Bmatrix}, \quad \begin{Bmatrix} \gamma_{xz} \\ \gamma_{yz} \end{Bmatrix}^{(0)} = -\frac{1}{3c_1} \begin{Bmatrix} \gamma_{xz} \\ \gamma_{yz} \end{Bmatrix}^{(2)} = \begin{Bmatrix} \phi_x + \frac{\partial w}{\partial x} \\ \phi_y + \frac{\partial w}{\partial y} \end{Bmatrix}
 \end{aligned}$$

RLPT – Equations of Motion

$$\frac{\partial N_{xx}}{\partial x} + \frac{\partial N_{xy}}{\partial y} = I_0 \ddot{u}_0 + J_1 \ddot{\phi}_x - c_1 I_3 \frac{\partial \ddot{w}_0}{\partial x}$$

$$\frac{\partial N_{xy}}{\partial x} + \frac{\partial N_{yy}}{\partial y} = I_0 \ddot{v}_0 + J_1 \ddot{\phi}_y - c_1 I_3 \frac{\partial \ddot{w}_0}{\partial y}$$

Omit subscript '0'
from u , v , and w

$$\frac{\partial \bar{Q}_x}{\partial x} + \frac{\partial \bar{Q}_y}{\partial y} + \frac{\partial}{\partial x} \left(N_{xx} \frac{\partial w_0}{\partial x} + N_{xy} \frac{\partial w_0}{\partial y} \right) + \frac{\partial}{\partial y} \left(N_{xy} \frac{\partial w_0}{\partial x} + N_{yy} \frac{\partial w_0}{\partial y} \right)$$

$$+ c_1 \left(\frac{\partial^2 P_{xx}}{\partial x^2} + 2 \frac{\partial^2 P_{xy}}{\partial x \partial y} + \frac{\partial^2 P_{yy}}{\partial y^2} \right) + q = I_0 \ddot{w}_0 - c_1^2 I_6 \left(\frac{\partial^2 \ddot{w}_0}{\partial x^2} + \frac{\partial^2 \ddot{w}_0}{\partial y^2} \right)$$

$$+ c_1 \left[I_3 \left(\frac{\partial \ddot{u}_0}{\partial x} + \frac{\partial \ddot{v}_0}{\partial y} \right) + J_4 \left(\frac{\partial \ddot{\phi}_x}{\partial x} + \frac{\partial \ddot{\phi}_y}{\partial y} \right) \right]$$

$$\frac{\partial \bar{M}_{xx}}{\partial x} + \frac{\partial \bar{M}_{xy}}{\partial y} - \bar{Q}_x = J_1 \ddot{u}_0 + K_2 \ddot{\phi}_x - c_1 J_4 \frac{\partial \ddot{w}_0}{\partial x}$$

$$\frac{\partial \bar{M}_{xy}}{\partial x} + \frac{\partial \bar{M}_{yy}}{\partial y} - \bar{Q}_y = J_1 \ddot{v}_0 + K_2 \ddot{\phi}_y - c_1 J_4 \frac{\partial \ddot{w}_0}{\partial y}$$

$$\bar{M}_{\alpha\beta} = M_{\alpha\beta} - c_1 P_{\alpha\beta}, \quad \bar{Q}_\alpha = Q_\alpha - c_2 R_\alpha$$

RLPT – Definition of Stress Resultants

Conventional Stress Resultants

$$\begin{Bmatrix} N_{xx} \\ N_{yy} \\ N_{xy} \end{Bmatrix} = \int_{-\frac{h}{2}}^{\frac{h}{2}} \begin{Bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{xy} \end{Bmatrix} dz, \quad \begin{Bmatrix} M_{xx} \\ M_{yy} \\ M_{xy} \end{Bmatrix} = \int_{-\frac{h}{2}}^{\frac{h}{2}} \begin{Bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{xy} \end{Bmatrix} z dz$$

Higher-Order Stress Resultants

$$\begin{Bmatrix} P_{xx} \\ P_{yy} \\ P_{xy} \end{Bmatrix} = \int_{-\frac{h}{2}}^{\frac{h}{2}} \begin{Bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{xy} \end{Bmatrix} z^3 dz, \quad \begin{Bmatrix} R_x \\ R_y \end{Bmatrix} = \int_{-h/2}^{h/2} \begin{Bmatrix} \sigma_{yz} \\ \sigma_{xz} \end{Bmatrix} z^2 dz$$

Mass Inertias

$$I_i = \int_{-\frac{h}{2}}^{\frac{h}{2}} \rho (z)^i dz \quad (i = 0, 1, 2, \dots, 6)$$

$$J_i = I_i - c_1 I_{i+2}, \quad K_2 = I_2 - 2c_1 I_4 + c_1^2 I_6$$

RLPT – Boundary Conditions

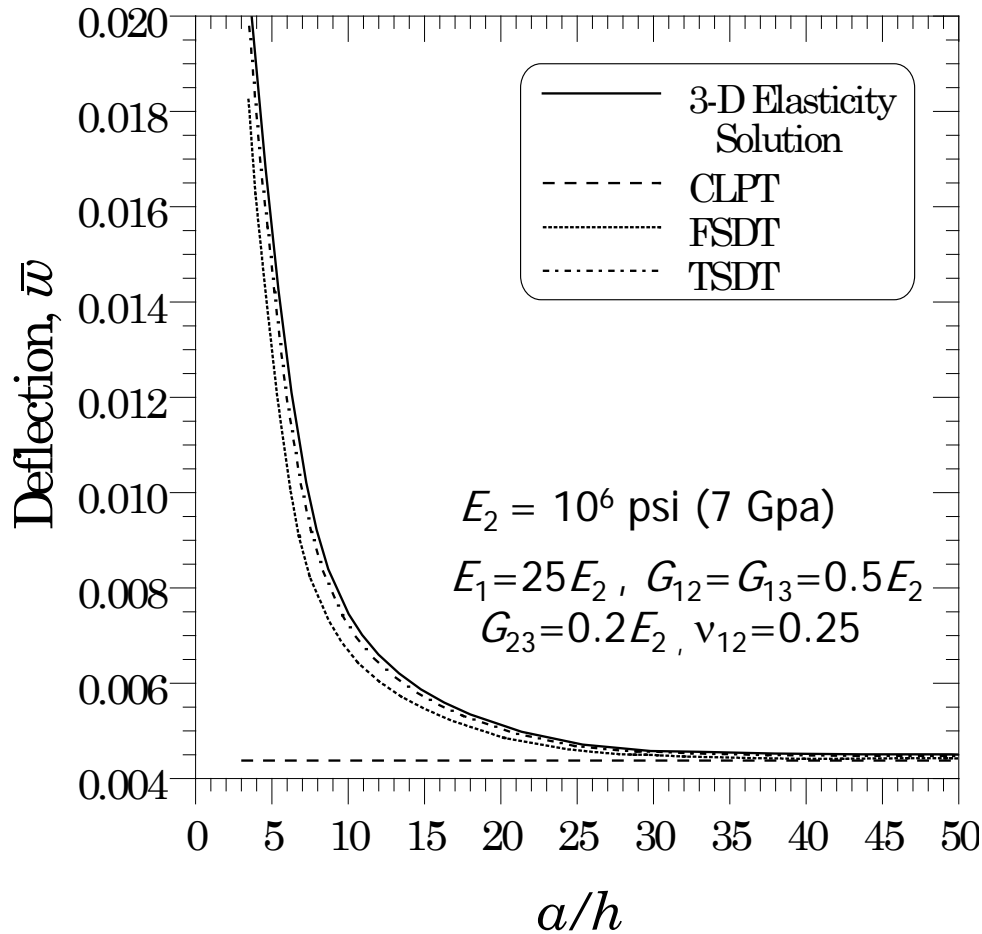
Primary Variables : $u_n, u_s, w_0, \frac{\partial w_0}{\partial n}, \phi_n, \phi_s$

Secondary Variables : $N_{nn}, N_{ns}, \bar{V}_n, P_{nn}, \bar{M}_{nn}, \bar{M}_{ns}$

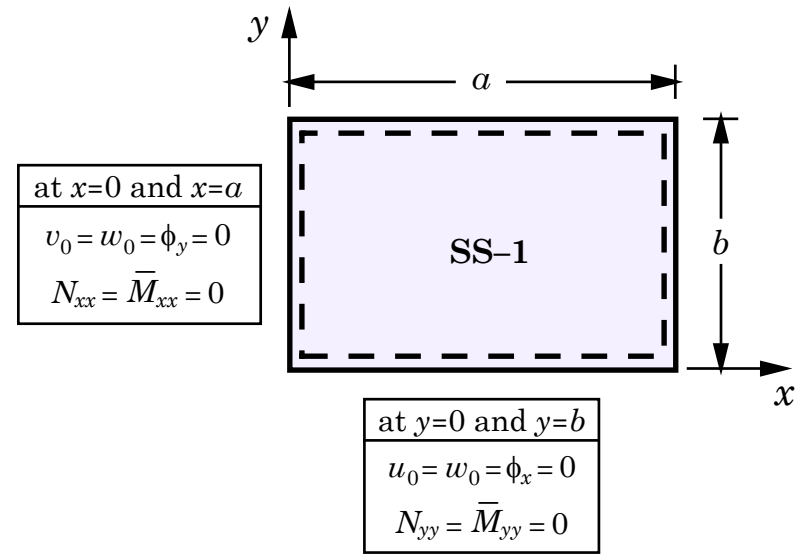
$$\begin{aligned} \bar{V}_n \equiv & c_1 \left[\left(\frac{\partial P_{xx}}{\partial x} + \frac{\partial P_{xy}}{\partial y} \right) n_x + \left(\frac{\partial P_{xy}}{\partial x} + \frac{\partial P_{yy}}{\partial y} \right) n_y \right] \\ & - c_1 \left[\left(I_3 \ddot{u}_0 + J_4 \ddot{\phi}_x - c_1 I_6 \frac{\partial \ddot{w}_0}{\partial x} \right) n_x + \left(I_3 \ddot{v}_0 + J_4 \ddot{\phi}_y - c_1 I_6 \frac{\partial \ddot{w}_0}{\partial y} \right) n_y \right] \\ & + (\bar{Q}_x n_x + \bar{Q}_y n_y) + \mathcal{P}(w_0) + c_1 \frac{\partial P_{ns}}{\partial s} \end{aligned}$$

$$\mathcal{P}(w_0) = \left(N_{xx} \frac{\partial w_0}{\partial x} + N_{xy} \frac{\partial w_0}{\partial y} \right) n_x + \left(N_{xy} \frac{\partial w_0}{\partial x} + N_{yy} \frac{\partial w_0}{\partial y} \right) n_y$$

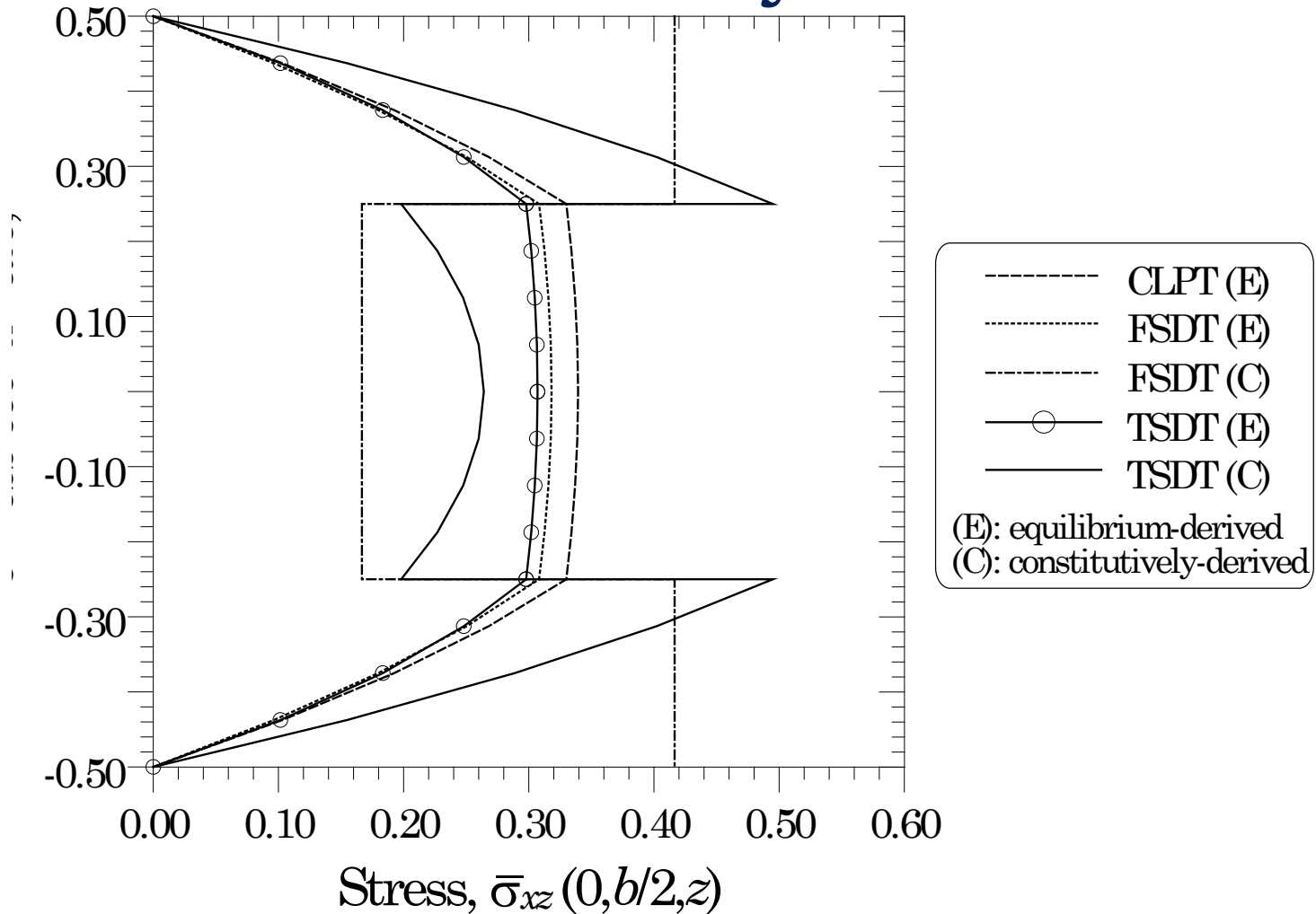
Bending of a symmetric cross-ply (0/90/90/0) laminate under uniformly distributed load



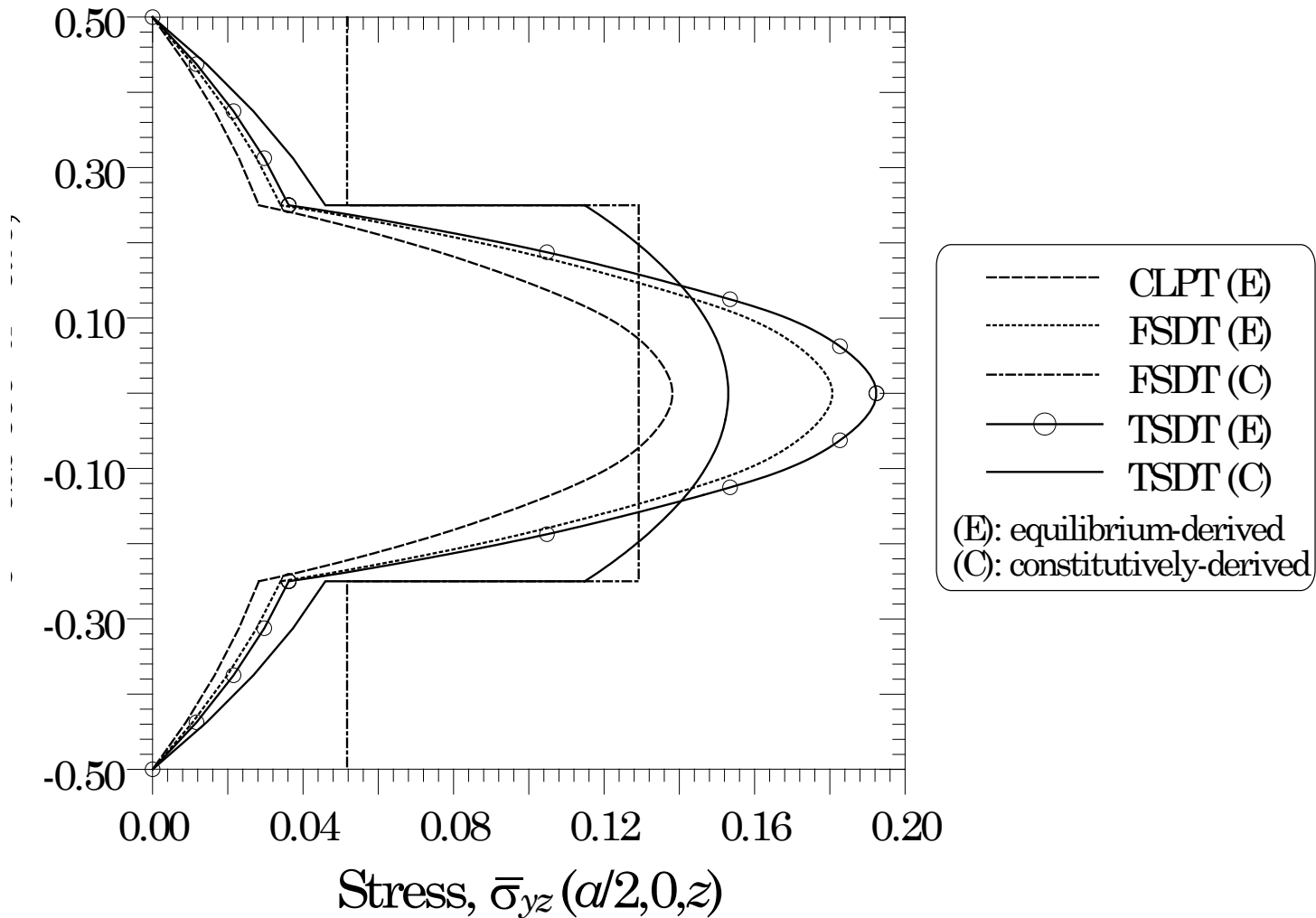
SS-1 Boundary Conditions



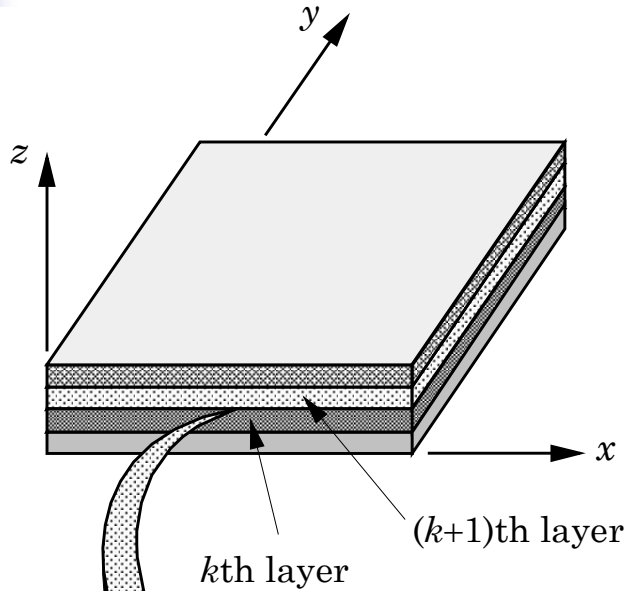
Bending of a symmetric cross-ply (0/90/90/0) laminate under uniformly distributed load



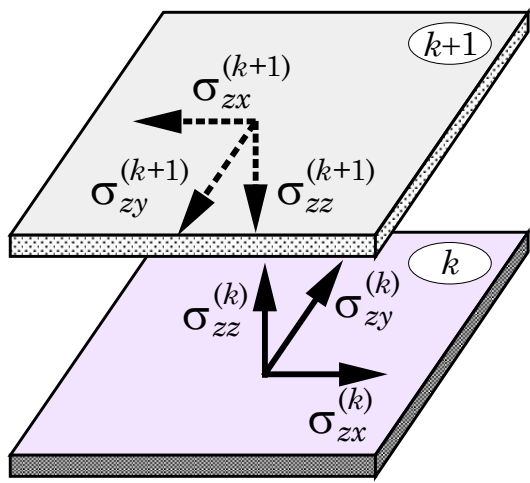
Bending of a symmetric cross-ply (0/90/90/0) laminate under uniformly distributed load



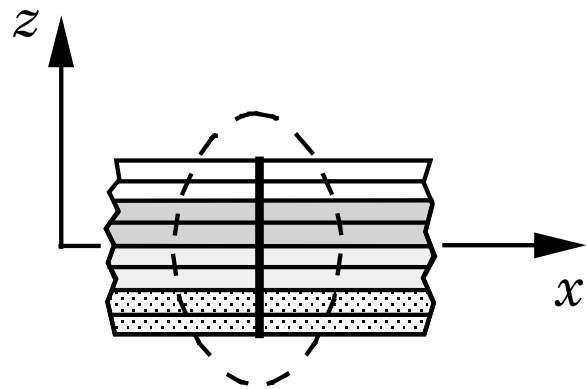
LAYERWISE LAMINATE THEORY



Equilibrium of Interlaminar Stresses



$$\begin{aligned} \sigma_{zx}^{(k+1)} &= \sigma_{zx}^{(k)} \\ \sigma_{zy}^{(k+1)} &= \sigma_{zy}^{(k)} \\ \sigma_{zz}^{(k+1)} &= \sigma_{zz}^{(k)} \end{aligned}$$



INTERLAMINAR STRESS AND STRAIN CONTINUITY

Equilibrium Requirements

$$\begin{Bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{xy} \end{Bmatrix}^{(k)} \neq \begin{Bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{xy} \end{Bmatrix}^{(k+1)}, \quad \begin{Bmatrix} \sigma_{xz} \\ \sigma_{yz} \\ \sigma_{zz} \end{Bmatrix}^{(k)} = \begin{Bmatrix} \sigma_{xz} \\ \sigma_{yz} \\ \sigma_{zz} \end{Bmatrix}^{(k+1)}$$

$$\begin{Bmatrix} \sigma_{xz} \\ \sigma_{yz} \\ \sigma_{zz} \end{Bmatrix}^{(k)} = \begin{Bmatrix} \sigma_{xz} \\ \sigma_{yz} \\ \sigma_{zz} \end{Bmatrix}^{(k+1)} \rightarrow \begin{Bmatrix} \gamma_{xz} \\ \gamma_{yz} \\ \varepsilon_{zz} \end{Bmatrix}^{(k)} \neq \begin{Bmatrix} \gamma_{xz} \\ \gamma_{yz} \\ \varepsilon_{zz} \end{Bmatrix}^{(k+1)}$$

Single-Layer Theories

$$\begin{Bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{xy} \end{Bmatrix}^{(k)} \neq \begin{Bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{xy} \end{Bmatrix}^{(k+1)}, \quad \begin{Bmatrix} \sigma_{xz} \\ \sigma_{yz} \\ \sigma_{zz} \end{Bmatrix}^{(k)} \neq \begin{Bmatrix} \sigma_{xz} \\ \sigma_{yz} \\ \sigma_{zz} \end{Bmatrix}^{(k+1)}$$

$$\begin{Bmatrix} \varepsilon_{xx} \\ \varepsilon_{yy} \\ \gamma_{xy} \end{Bmatrix}^{(k)} = \begin{Bmatrix} \varepsilon_{xx} \\ \varepsilon_{yy} \\ \gamma_{xy} \end{Bmatrix}^{(k+1)}, \quad \begin{Bmatrix} \gamma_{xz} \\ \gamma_{yz} \\ \varepsilon_{zz} \end{Bmatrix}^{(k)} = \begin{Bmatrix} \gamma_{xz} \\ \gamma_{yz} \\ \varepsilon_{zz} \end{Bmatrix}^{(k+1)}$$

Layerwise Displacement Field, Governing Equations, and FEM Approximation

$$u(x, y, z, t) = \sum_{I=1}^N U_I(x, y, t) \Phi^I(z)$$

$$v(x, y, z, t) = \sum_{I=1}^N V_I(x, y, t) \Phi^I(z)$$

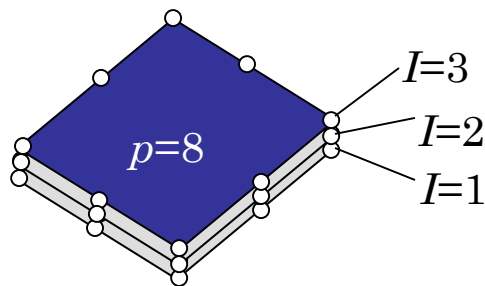
$$w(x, y, z, t) = \sum_{I=1}^M W_I(x, y, t) \Psi^I(z)$$

$$\frac{\partial N_{xx}^I}{\partial x} + \frac{\partial N_{xy}^I}{\partial y} - Q_x^I = \sum_{J=1}^N I^{IJ} \frac{\partial^2 U_J}{\partial t^2}$$

$$\frac{\partial N_{xy}^I}{\partial x} + \frac{\partial N_{yy}^I}{\partial y} - Q_y^I = \sum_{J=1}^N I^{IJ} \frac{\partial^2 V_J}{\partial t^2}$$

$$\frac{\partial \tilde{Q}_x^I}{\partial x} + \frac{\partial \tilde{Q}_y^I}{\partial y} - \tilde{Q}_z^I + \tilde{N}^I + q_b \delta_{I1} + q_t \delta_{IM} = \sum_{J=1}^M \tilde{I}^{IJ} \frac{\partial^2 W_J}{\partial t^2}$$

Finite element approximation



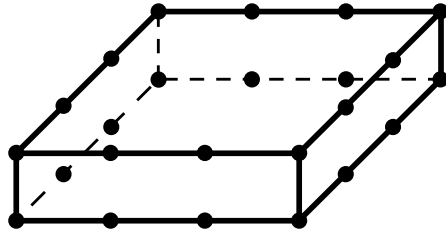
$$U_I(x, y, t) = \sum_{j=1}^p U_I^j(t) \psi_j(x, y)$$

$$V_I(x, y, t) = \sum_{j=1}^p V_I^j(t) \psi_j(x, y)$$

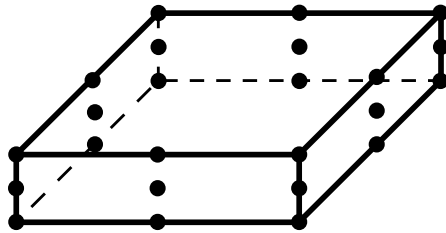
$$W_I(x, y, t) = \sum_{j=1}^q W_I^j(t) \varphi_j(x, y)$$

Layerwise Kinematic Model

Conventional 3D

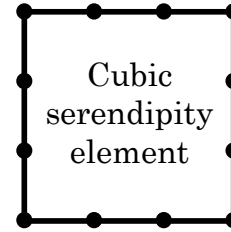


(1a)

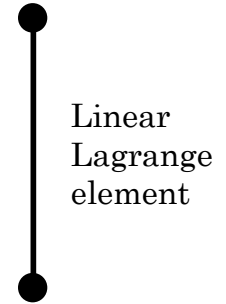


(2a)

Layerwise 2D + 1D

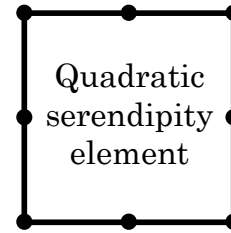


(in-plane)

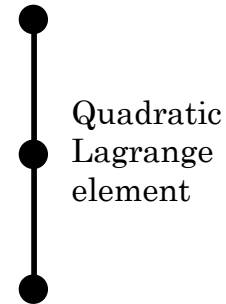


(through thickness)

(1b)



(in-plane)



(through thickness)

(2b)

Table: Comparison of the number of operations needed to form the element stiffness matrices for equivalent elements in the conventional 3-D format and the layerwise 2-D format. Full quadrature is used in all.

Element Type [†]	Multipli.	Addition	Assignments
1a (3-D)	1,116,000	677,000	511,000
1b (LWPT)	423,000	370,000	106,000
2a (3-D)	1,182,000	819,000	374,000
2b (LWPT)	284,000	270,000	69,000

[†] *Element 1a*: 72 degrees of freedom, 24-node 3-D isoparametric hexahedron with cubic in-plane interpolation and linear transverse interpolation.

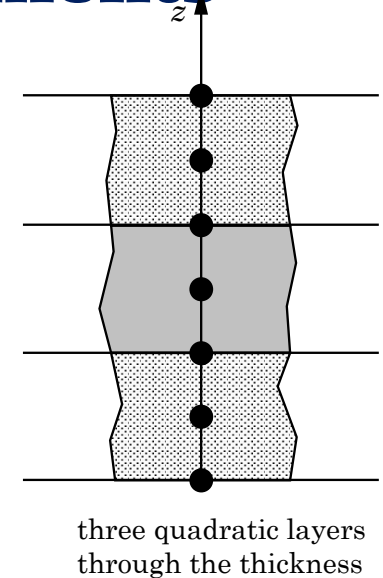
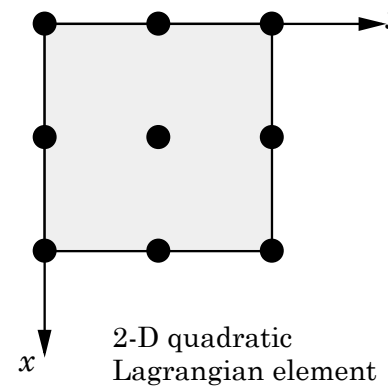
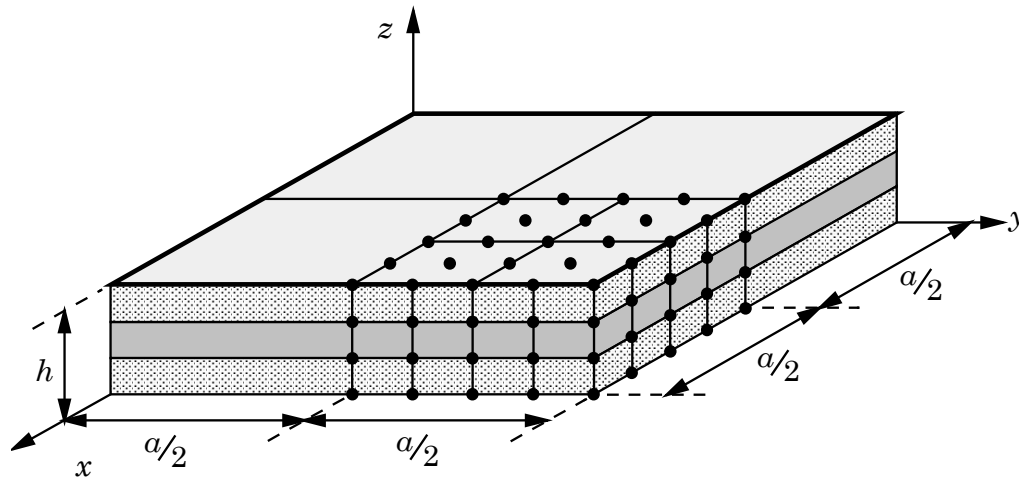
Element 1b: 72 degrees of freedom, E12–L1 layerwise element.

Element 2a: 81 degrees of freedom, 27-node 3-D isoparametric hexahedron with quadratic interpolation in all three directions.

Element 2b: 81 degrees of freedom, E9–Q1 layerwise element.

Layerwise Kinematic Model

3D modeling with 2D & 1D elements



$$E_1 = 25 \times 10^6 \text{ psi}, \quad E_2 = E_3 = 10^6 \text{ psi}$$

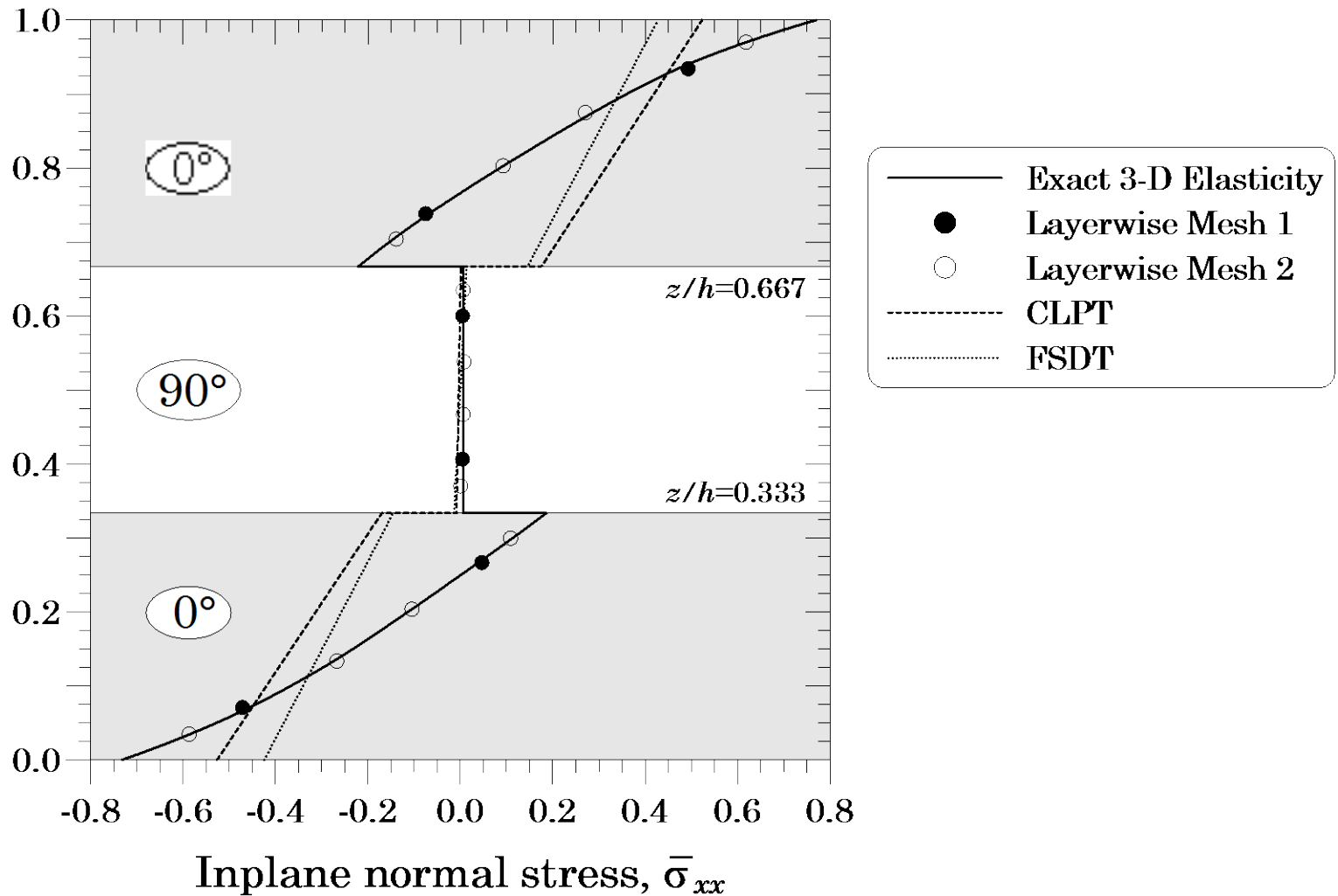
$$G_{12} = 0.5 \times 10^6 \text{ psi}, \quad G_{13} = G_{23} = 0.2 \times 10^6 \text{ psi}, \quad \nu_{12} = \nu_{13} = \nu_{23} = 0.25$$

$$u(x, a/2, z) = u(a/2, y, z) = 0$$

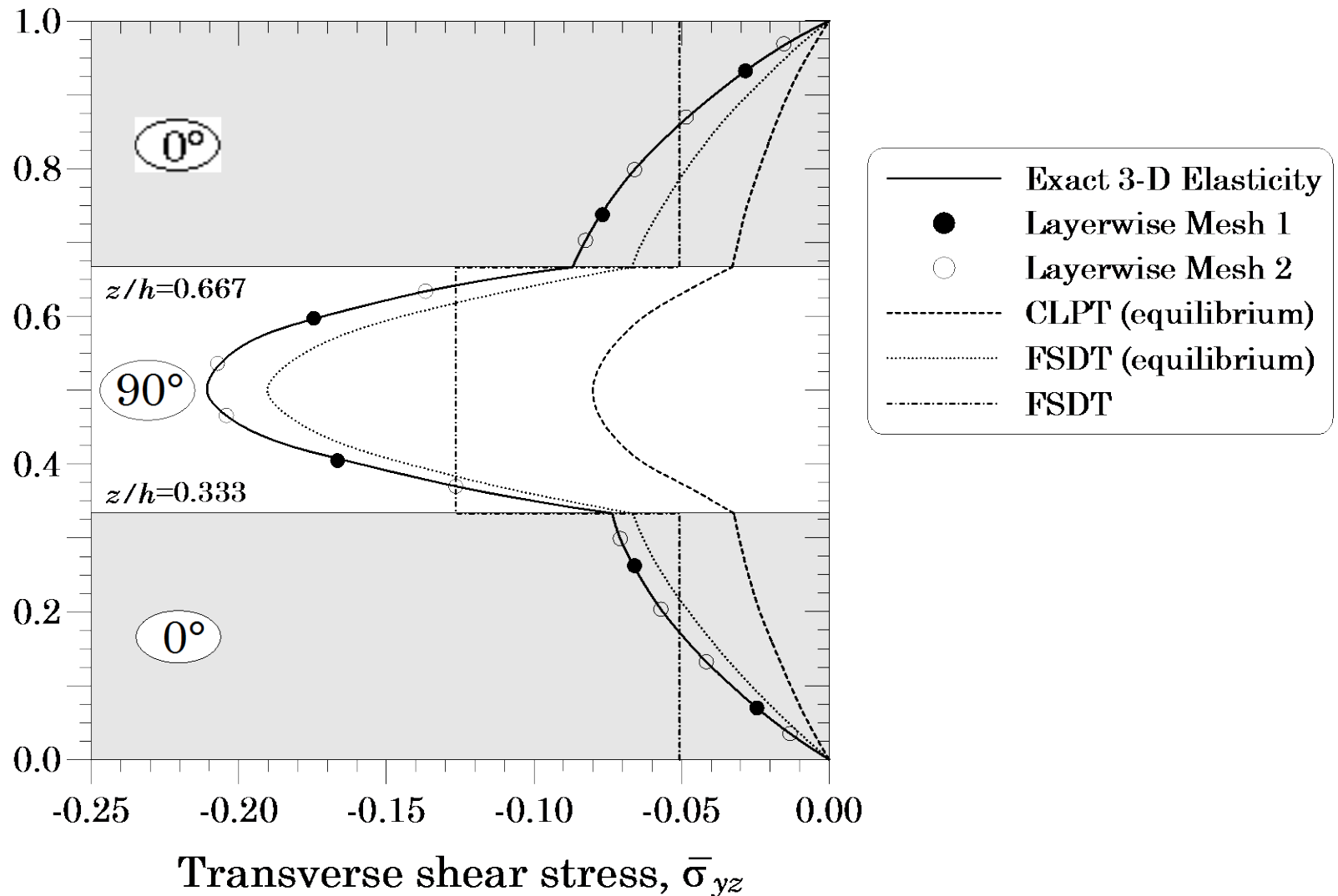
$$v(a/2, y, z) = u(x, a/2, z) = 0$$

$$\text{J.N. Reddy} \quad w(x, a, z) = u(a, y, z) = 0$$

Validation of the Layerwise Theory



Verification of the Layerwise Theory





Variable Kinematic Model for Global-Local Analysis

Composite displacement field:

$$u_i(x, y, z) = u_i^{ESL}(x, y, z) + u_i^{LWT}(x, y, z)$$

ESL Displacement field:

$$u_1^{ESL}(x, y, z) = u_0(x, y) + z\phi_x(x, y)$$

$$u_2^{ESL}(x, y, z) = v_0(x, y) + z\phi_y(x, y)$$

$$u_3^{ESL}(x, y, z) = w_0(x, y)$$

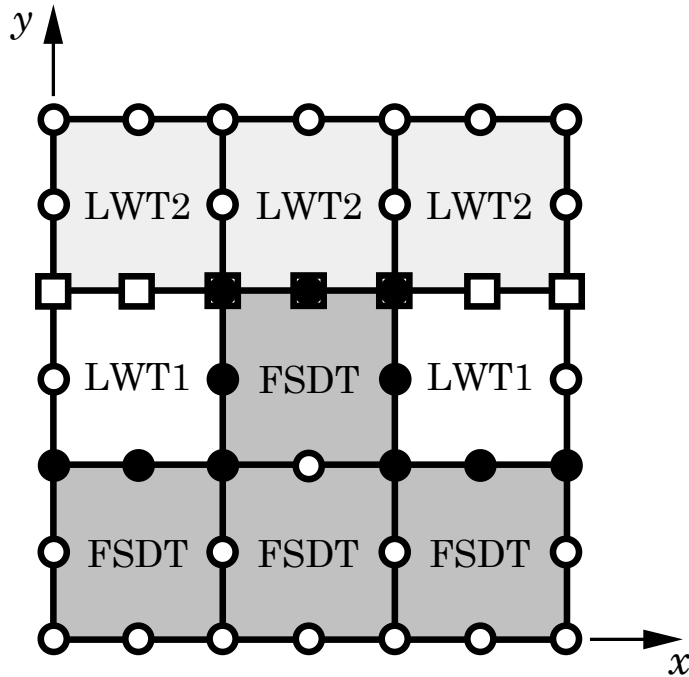
LWT Displacement field:

$$u_1^{LWT}(x, y, z) = \sum_{I=1}^N U_I(x, y)\Phi^I(z)$$

$$u_2^{LWT}(x, y, z) = \sum_{I=1}^N V_I(x, y)\Phi^I(z)$$

$$u_3^{LWT}(x, y, z) = \sum_{I=1}^M W_I(x, y)\Psi^I(z)$$

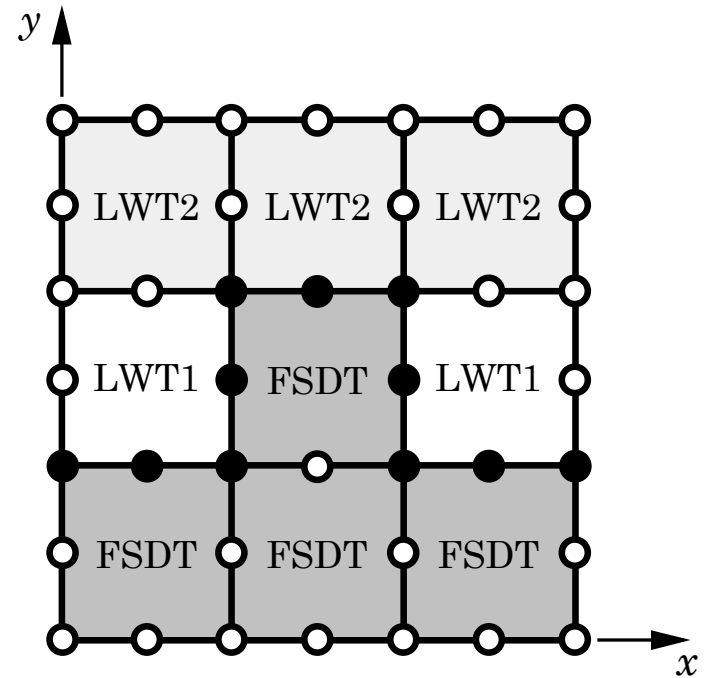
Sub-region Continuity of the Solution



At nodes ●, set $U_j = V_j = 0, j=1,2,\dots,n$

At nodes ■, set $W_j = 0, j=1,2,\dots,n$

(a) Enforcing **strict** subregion compatibility



At nodes ●, set $U_j = V_j = 0, j=1,2,\dots,n$

(b) Enforcing **relaxed** subregion compatibility

Free-Edge Problem

$$E_1 = 20 \times 10^6 \text{ psi.}$$

$$E_2 = 2.1 \times 10^6 \text{ psi.}$$

$$E_3 = 2.1 \times 10^6 \text{ psi.}$$

$$G_{12} = 0.85 \times 10^6 \text{ psi.}$$

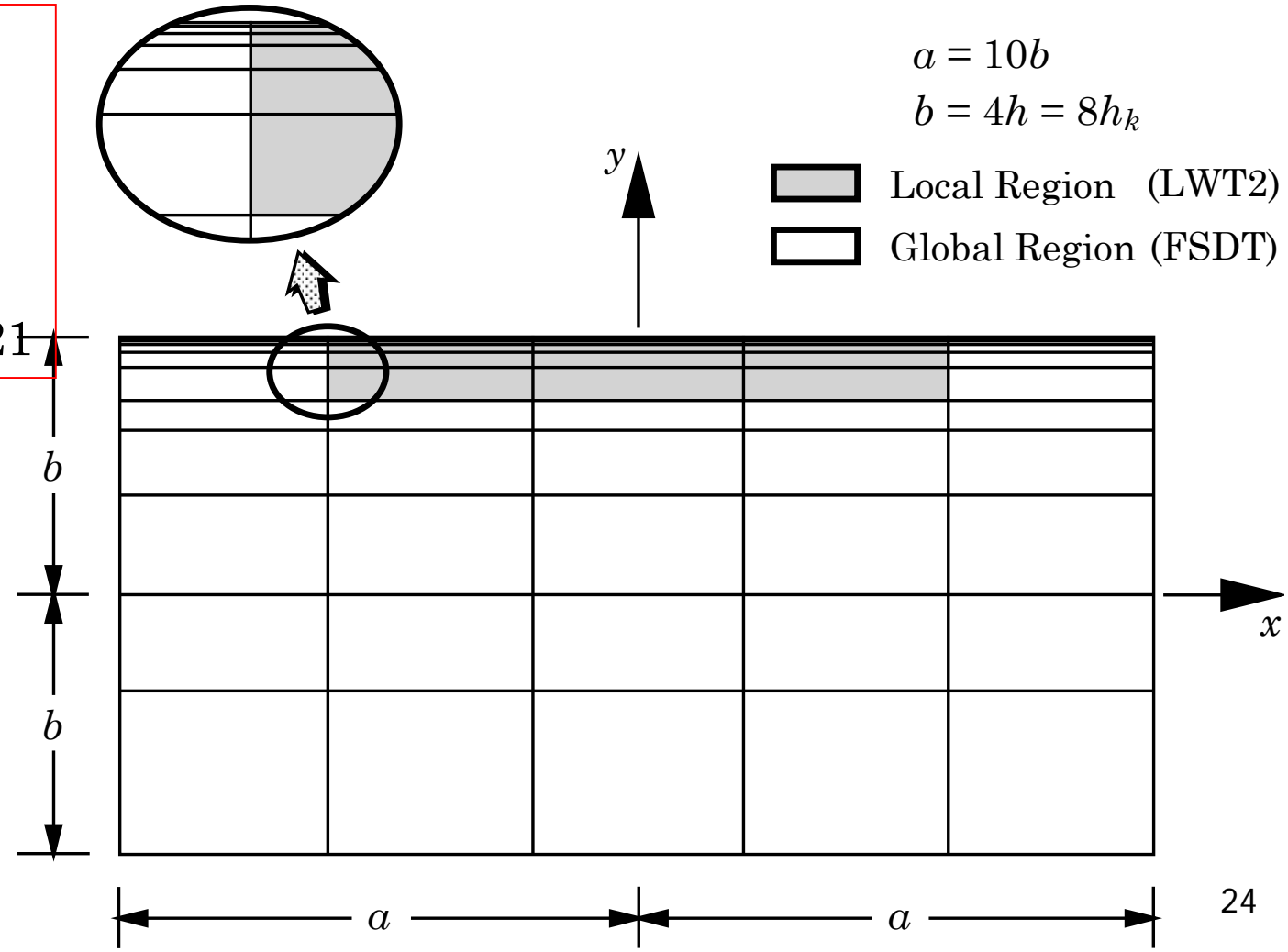
$$G_{13} = G_{23} = G_{12}$$

$$\nu_{13} = \nu_{23} = \nu_{12} = 0.21$$

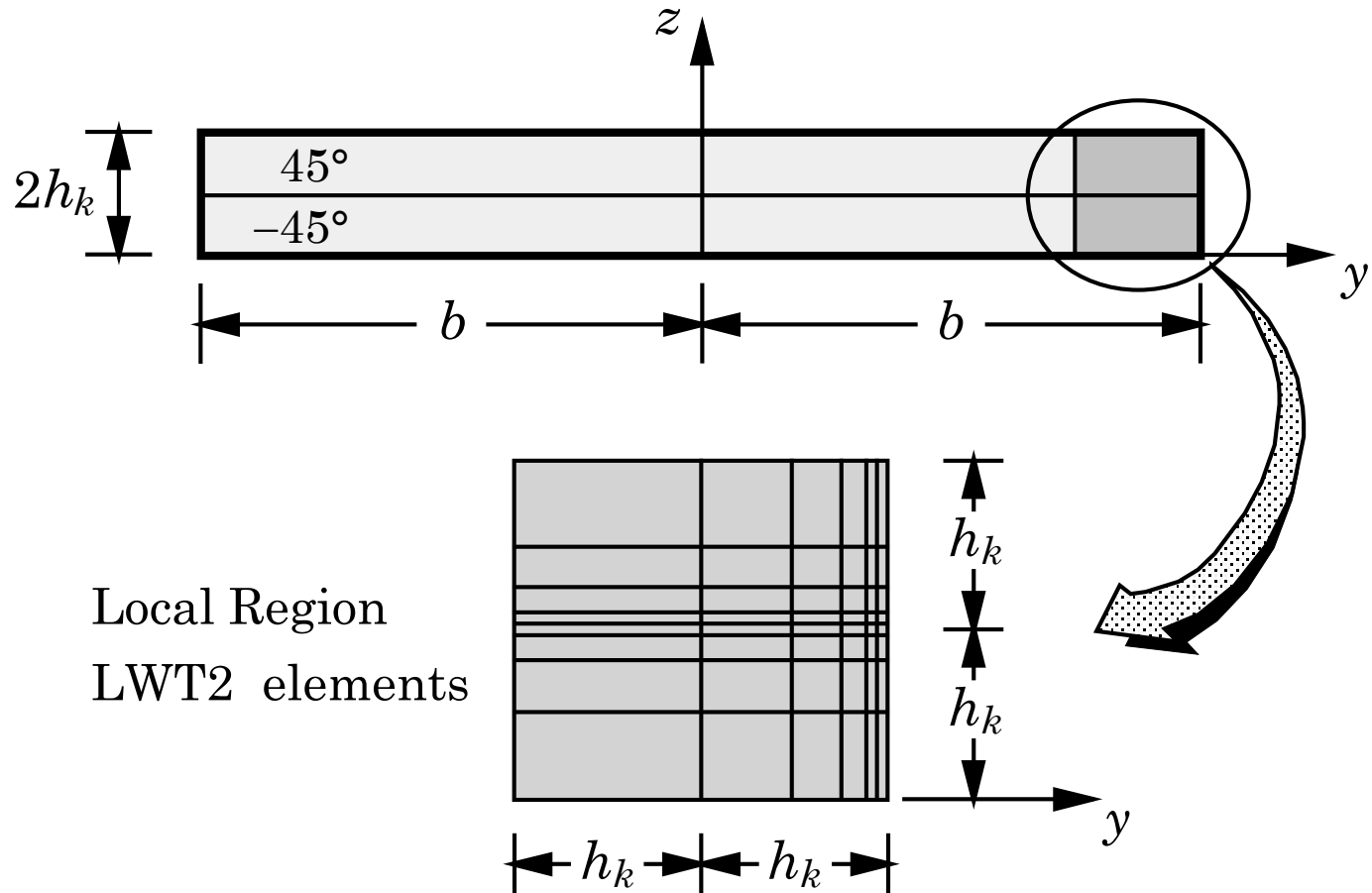
$$a = 10b$$

$$b = 4h = 8h_k$$

-  Local Region (LWT2)
-  Global Region (FSDT)



Free-Edge Problem (continued)



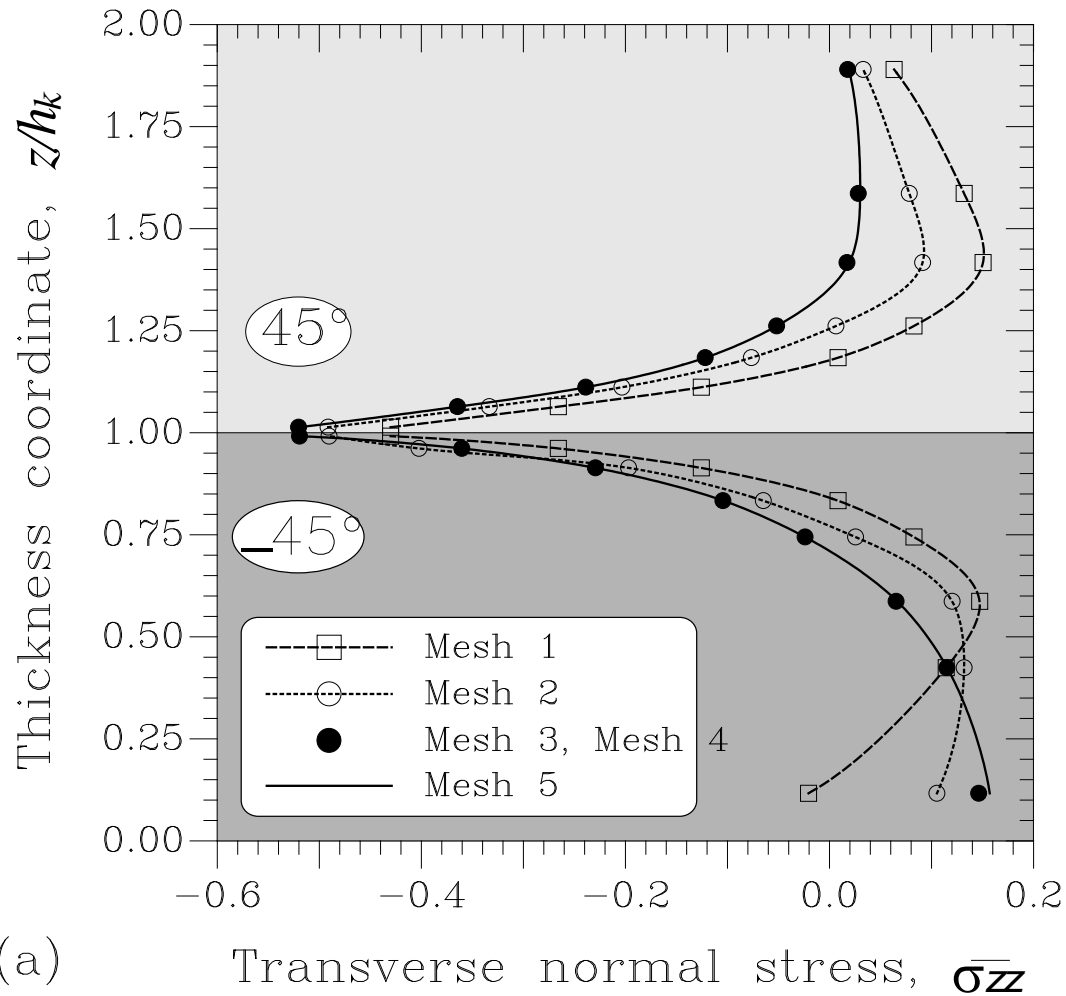
Free-Edge Problem (continued)

Table: Description of global-local meshes for the $(45/-45)_s$ laminate under axial extension.

Remarks	Mesh 1	Mesh 2	Mesh 3	Mesh 4	Mesh 5
Number of Elements in Local LWT2 Region	3×4	3×5	3×6	3×7	5×11
Width of Local Region	$\frac{1}{2}h_k$	h_k	$2h_k$	$3h_k$	$16h_k$
Length of Local Region	$\frac{6}{5}a$	$\frac{6}{5}a$	$\frac{6}{5}a$	$\frac{6}{5}a$	$2a$
Total Number of Active D.O.F. in VKFE Mesh (Strict Compatibility)	1,986	2,400	2,814	3,228	9,116
Total Number of Active D.O.F. in VKFE Mesh (Relaxed Compatibility)	2,354	2,800	3,246	3,690	9,116

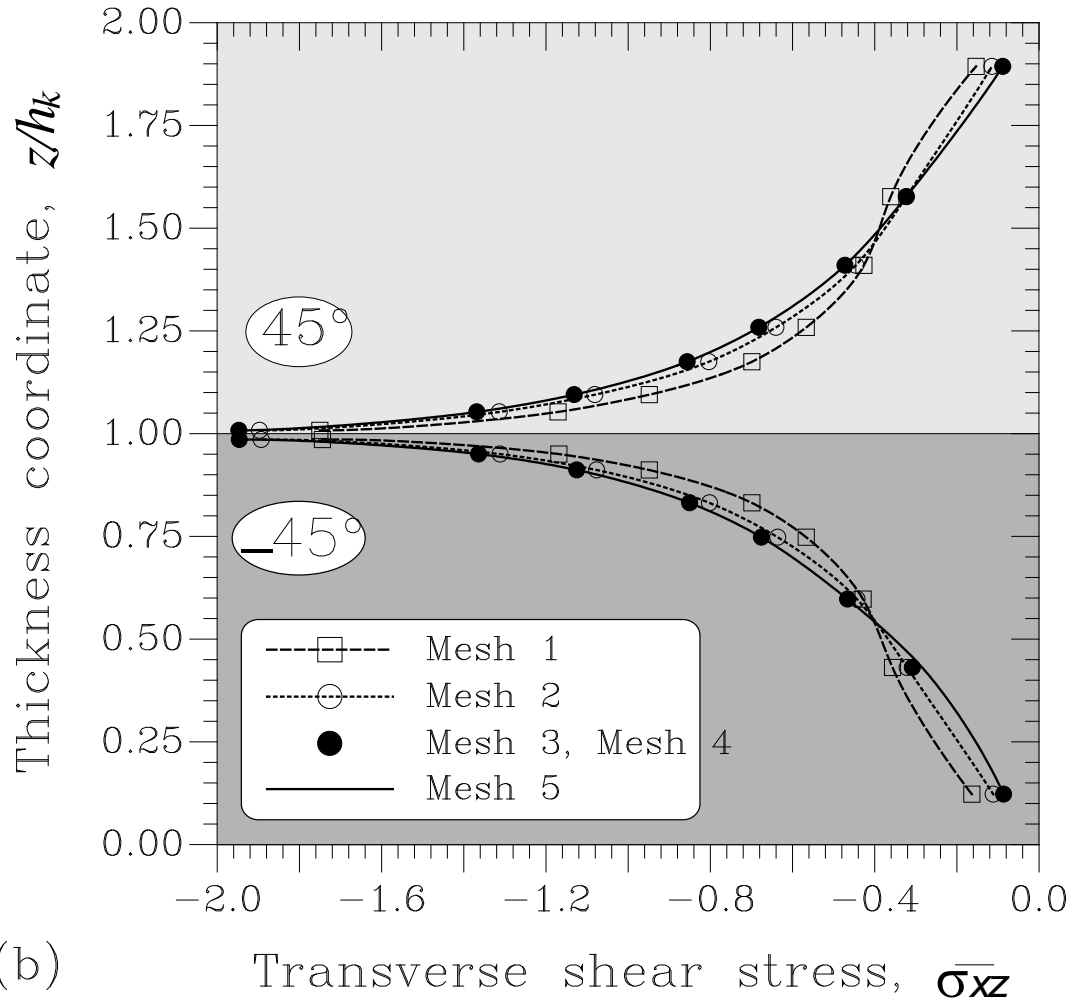
h_k = thickness of a single material ply. All five VKFE meshes have the exact same in-plane discretization (5×11).

Free-Edge Problem (continued)



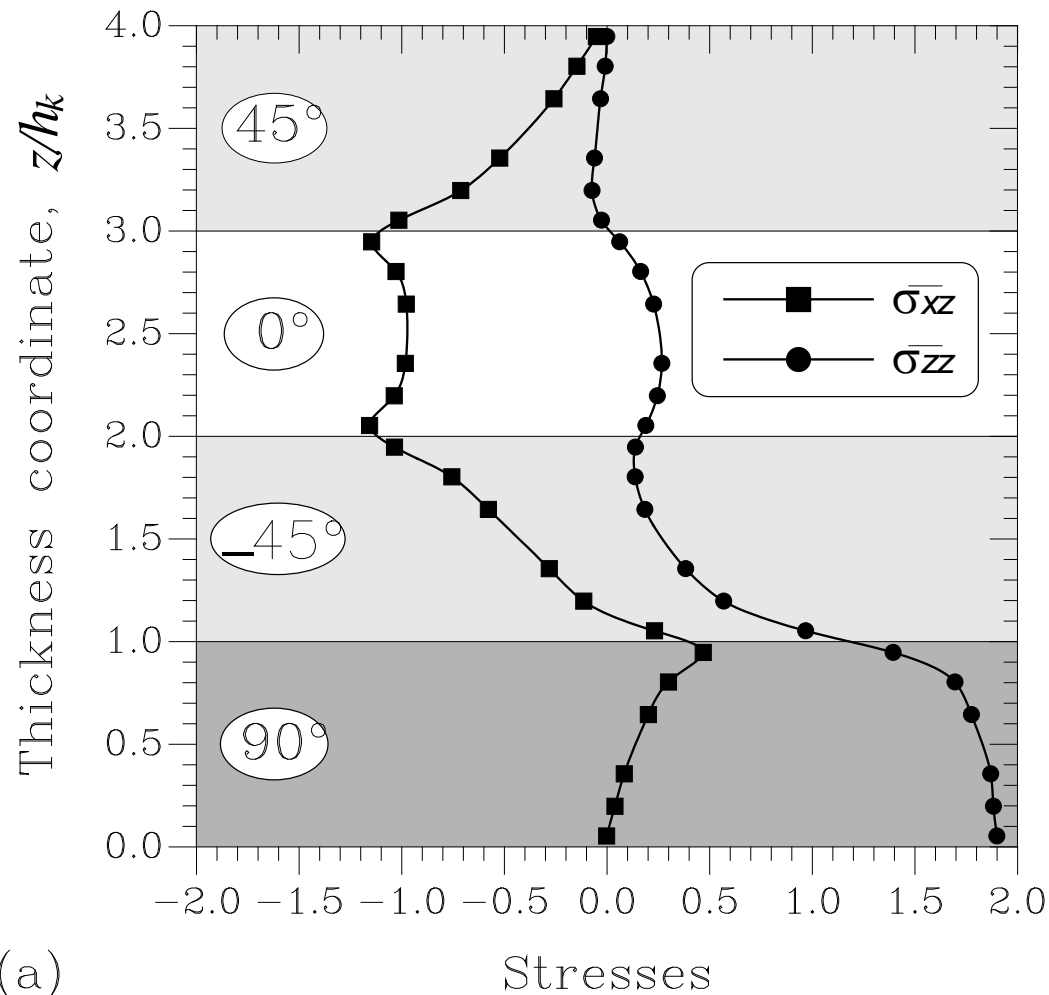
(a)

Free-Edge Problem (continued)



(b)

Free-Edge Problem (continued)



(a)



Summary

In this lecture, we have discussed the following topics:

Third-order Shear Deformation Plate Theory

Development of governing equations

Numerical results

Layerwise Laminate Theory

Development of governing equations

Global-local analysis

Numerical results